TRANSIENT, LAMINAR, FREE-FORCED CONVECTION WITH HEAT AND MASS TRANSFER FROM A VERTICAL ISOTHERMAL PLATE

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DEPARTMENT OF MECHANICAL ENGINEERING
INDIAN INSTITUTE OF TECHNOLOGY, KANPUR
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By ULHAS R. JAGDHANE

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CERTIFICATE

CERTIFIED that the thesis titled, "TRANSIENT, LAMINAR, FREE-FORCED CONVECTION WITH HEAT AND MASS TRANSFER FROM A VERTICAL ISOTHERMAL PLATE" has been submitted by Ulhas R-Jagdhane under my supervision and that this work has not been submitted elsewhere for award of a degree.

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Ulhas R. Jagdhane

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NOM ENCLATURE

С	concentration
С	dimensionless concentration, $(c-c_{\infty})/(c_{W}-c_{\infty})$
C _p	specific heat of the fluid at constant pres
c'''	rate of species generation per unit volume
D	chemical molecular diffusivity
е	specific internal energy of the mixture
g	acceleration due to gravity
Gr	thermal Grashof number, $\beta g L^3 (T_W^- T_\infty)/\nu^2$
G r*	mass Grashof number, $\beta^* gL^3(c_W^-c_\infty^-)/\nu^2$
h	local heat-transfer coefficient, -K(3T/3Y)
h_{D}	local mass-transfer coefficient, -D(3c/3Y)
ħ	average heat transfer coefficient, $\frac{1}{L} \int_{0}^{L} h dx$
$ar{ t h}_{ extsf{D}}$	average mass-transfer coefficient, $\frac{1}{L} \int_{0}^{L} h_{D} dx$
i	represents time
j	represents location in X-direction
k	represents location in Y-direction
K	thermal conductivity of the fluid
L	length of the flat plate
m	vertical mass flux rate
И	buoyancy ratio parameter, $\beta * (c_W - c_\infty) / \beta (T_W - T_\infty)$
27	
^{Nu} x	instantaneous local Nusselt number, $\frac{hx}{K}$
^{Nu} x ^{Nu} m	instantaneous local Nusselt number, $\frac{hx}{K}$ instantaneous mean Nusselt number, $\frac{hL}{K}$ Gr ⁻¹ /

đ.	heat flux rate
d,w	rate of energy generation per unit volume
Sc	Schmidt number, v/D
Sh _x	instantaneous local Sherwood number, $\frac{h_D x}{D}$
Sh _m	instantaneous mean Sherwood number, $\frac{\bar{h}_DL}{D}$ Cs-1/4
t	t i me
T	temperature
u	x-velocity component
U .	dimensionless X-velocity component, $\frac{uL}{v}$ Gr ^{-1/2}
v	y-velocity component
V	dimensionless Y-velocity component, $\frac{vL}{v}$ Gr ^{-1/4}
$\vec{\nabla}$	velocity vector
\mathbf{x}	spatial coordinate along the plate
x	dimensionaless spatial coordinate along the plate,
У	spatial coordinate normal to the plate
Y	dimensionless spatial coordinate normal to the
	plate, T Gr1/4
α	thermal diffusivity, K/pC
α _k , α' _k , α'' _k	lower diagonal elements in tridiagonal matrices for
	momentum, energy, and species equations respective
β	volumetric coefficient of thermal expansion
β *	volumetric coefficient of expansion with
	concentration ·
βκ'βκ'βκ	main diagonal elements in tridiagonal matrices
	for momentum, energy, and species equations
	respectively
ô	velocity boundary layer thickness

```
concentration boundary layer thickness
ôح
               thermal boundary layer thickness
\delta_{+}
Δτ
               dimensionless time-step
\Delta X
               dimensionless step size in X-direction
\Delta Y
               dimensionless step-sixe in Y-direction
               dimensionless temperature, (T-T_{\infty})/(T_{w}-T_{\infty})
θ
               relaxation factor
λ
               dynamic viscosity of the fluid
               kinematic viscosity of the fluid, \mu/\rho
               dimensionless time, \frac{tr}{\tau^2} Gr<sup>1/2</sup>
τ
               density of the fluid
ρ
               function associated with the dissipation of energy
               ratio of step sizes
               elements of known right hand side column vectors
               of momentum, energy and species equations respectively
Qk, Qk, Qk
               upper diagonal elements in tridiagonal matrices
               for momentum, energy, and species equation respectively
               notation used for (\frac{\partial \theta}{\partial Y}) \Big|_{Y=0}
ځ
               notation used for -(\frac{\partial C}{\partial Y})\Big|_{Y=0}
ζ
Subscripts
C
               based on species concentration
U
               based on velocity
               at the surface of the plate
W
               based on the distance from the leading edge of the
X
               plate
6
               based on temperature
               free stream conditions
Superscript
```

iteration number

1

Chapter 1

INTRODUCTION

There are many transport processes which occur in nature and in man-made devices in which flow arises simply due to the gradientsof density, temperatures, and/or chemical composition in a body force field, such as the gravitational field. Ever since the pioneering efforts by Lorenz [1] in 1831, such processes have been of considerable interest to engineers and scientists because of their numerous applications.

Processes in which buoyancy as the driving force arises solely due to the temperature difference have received considerable attention for both steady and transient, internal and external, and laminar and turbulent flows with several additional conditions and effects such as combined free and forced convection, etc. However, buoyancy effects resulting from concentration gradients in multicomponent mixtures can be just as important in generating fluid motion as the temperature gradients, as pointed out by Gebhart and Pera [2]. Fields of interest in which combined heat and mass transfer, under the condition of free convection are frequently concountered are: the evaporation of water from the surface of a water body in the absence of strong winds, as from ponds and lakes; drying processes in nature; distribution of temperature and moisture over agricultural fields and groves

of fruit trees; damage of crops due to freezing; formation and dispersion of fog; pollution of the environment; technological applications such as design of chemical processing equipment, etc.

In a large number of important applications, however, the convective process is neither predominantly natural nor predominantly forced; both modes being significant. The question then is whether the forced convection masks the natural convection, and if so, under what conditions? The answer to this question is provided here.

Consideration of transient natural convection is also important in many technological applications, since the heat transfer rates vary considerably during the transient stage. For given energy inputs this may result in over-heating and in consequent damage to various components of the systems, furnaces, electronic systems etc, which have, therefore to be designed to withstand the transients during the start up and shut down operations. We therefore consider the unsteady natural and forced convection in the presence of temperature and concentration gradients over a vertical flat plate. The flat plate provides the simplest geometry so that effects other than geometrical can be isolated.

Earlier developments:

One of the earliest studies with combined heat and mass transfer known to us is that of Somers [3]. It is concerned

with the combined thermal and species diffusion driven flow that would arise adjacent to a wetted isothermal vertical surface in a non-saturated atmosphere. The condition of very small diffusing species concentration was used and an integral method analysis was carried out for uniform surface temperature and uniform diffusing species concentration. The principal results were a transport relation and the indication that a combined driving force might be written in which the species diffusion contribution is modified by the square root of the Lewis number, i.e, Vie. The analysis is expected to be reasonable around Prandtl and Schmidt numbers of 1.0, with one buoyancy effect being very small compared to the other. Mathers et al. [4] formulated the same problem in terms of the boundary layer differential equations resulting from momentum, energy, and chemical species conservations at low concentration. Neglecting inertia effects the resulting equations were solved on an analogue computer for Pr = 1.0and Sc = 0.5 - 10.0 for ratio of species and thermal diffusion buoyancy effects of 1.0 and 0.5. The resultant transport information appears to support the VLe factor of Somers.

Possibility of similarity solutions for combined buoyancy effect flows formulated within the limitations assumed by Gill et al. [5] were considered by Lowell and Adams [6]. The only similarity solution found was that on an isothermal vertical surface. Results of numerical analysis of above similarity formulation were presented for a subliming organic

surface in air. Numerical difficulties that arose when buoyancy effects were opposed were noted and termed as flow instabilities, without any satisfactory justification.

Den Bounter [7] reports an experimental study of simultaneous thermal and chemical species diffusion by an electrochemical method between a vertical copper plate maintained at constant temperature and a copper sulphate-sulphuric acid solution. Measurements were made with the two buoyancy effects aiding and opposing each other. The mass and heat transfer parameters, correlated in terms of a combined buoyancy effect, calculated with the VIE term, agree well with a single curve for the effects aiding each other. However, for the two buoyancy effects opposing each other the disagreement from the single curve is random and over 30 percent in magnitude.

Bottemanne [8] has also considered steady state simultaneous heat and mass transfer along a vertical flat plate. Solution to the boundary layer equations was obtained only for Pr = 0.71 and Sc = 0.63. His theoretical solution agrees well with his experiments on heat transfer with simultaneous water evaporation into air.

The problem of combined forced and natural convection has been treated by Lloyd and Sparrow [9], covering conditions ranging from pure forced convection flow to combined flows with strong natural convection contribution, for an isothermal vertical flat plate. But buoyancy force arising only due to a

is employed and numerical results are presented for Pr values ranging from 0.003 to 100.

Some experimental work has also been carried out on mixed convection from a vertical surface but neglecting buoyancy effect due to concentration gradient. Kliegel [10] cmployed interferometric methods to determine the local heat transfer rates from a vertical surface located in an air stream. This data was found to be in very good agreement with the analytical results of Lloyd and Sparrow.

Over an isothermal vertical surface with aiding external flow and determined the local velocity and temperature profiles and the heat transfer rates. Good correlation between theory and experiment was obtained for the temperature profiles. The correlation was found to be very good for the heat transfer results, as expected from the agreement of temperature profiles. The limits of forced and free convection regions were also determined for Pr = 0.72.

Numerical solution for boundary layer equations for a transient free convection (buoyancy effects due to temperature gradient alone) over a vertical surface, subjected to a step change in the surface temperature, have been obtained by Hellums and Churchill [12]. The results converge to steady state values at large time and show a minimum in the Nusselt number during

the transient stage, as found earlier by Siegel.

Gebhart and Pera [2] studied laminar natural convection flows resulting from combined buovancy mechanisms over a vertical flat plate in terms of similarity solutions. Over a range of Schmidt numbers, both aiding and opposing buoyancy effects were considered for air and water and solutions were obtained. The results show many interesting effects on velocity, heat and mass transfer, and on laminar stability. study stratification is neglected and only power law variations, $t-t_{\infty} = N_{t}x^{n}$, $c-c_{\infty} = N_{c}x^{n}$ are considered in order to make the similarity solution work. Gebhart and Pera studied the problem for steady state. However, the added constraint, brought in because of the transient terms, does not allow problems of practical importance to be studied by the similarity variable method, its application being restricted to particular forms of the surface temperature distribution. A numerical integration method may be considered but the procedure is complex one and does not yield desired physical insight into the process.

More recently Callhan and Marner [13] have considered transient laminar free convection along a vertical, isothermal flat plate, arising from buoyancy forces created by both temperature and concentration gradients. The coupled nonlinear partial differential equations are solved numerically using an explicit finite difference scheme. Results were obtained for Pr = 1.0 and range of Schmidt numbers and for aiding mass

diffusion buoyancy forces (N > 0). Steady sta te local Nusselt and Sherwood numbers were compared with the results of Gebhart and Pera. Largest deviation was observed at X = 0.10 with a difference of 4.2 percent while excellent agreement was found at the trailing edge of the plate with a difference of only 0.75 percent.

Present Work:

The purpose of the present study is to investigate the problem of transient, laminar, combined forced and natural convection along an isothermal vertical plate which is subjected to a step-change in temperature and concentration. The study covers conditions ranging from pure natural convection to strong forced convection. The coupled nonlinear partial differential equations are solved numerically by a highly implicit finite difference procedure.

There are many interesting aspects of such flows, such as the resulting transport characteristics, the influence of combined buoyancy force effects and combined free and forced convection on the stability of the boundary layers, and the effects of the values of relative transport parameters, the Prandtl and Schmidt numbers. Of particular interest in this study are:

- i) The effect of the buoyancy forces due to mass transfer on the transient velocity profiles, temperature profiles, Nusselt number and Sherwood number.
- ii) The effect of free stream velocity U_∞ on the Nusselt and Sherwood numbers, and on the transient velocity, temperature and concentration profiles.

Chapter 2 ANALYSIS

2.1 Governing Equations:

Fundamental physical processes that occur in naturalconvection flows are essentially the same as those occuring in any fluid flow and diffusion processes. Therefore the basic equations used to interpret and analyse natural-convection flows are the same. There is however one fundamental difference between natural convection and forced convection. In natural convection fluid motion arises mainly from buoyancy and not from imposed motion or pressure difference. The buoyancy force arises due to the action of body force, usually gravity, on the density differences in a body of fluid, which results from temperature (and/or species-concentration) differences, which in turn are governed by the type of diffusion processs present. The diffusion processes which may be occuring simultaneously are coupled together resulting in much greater complexity and difficulty in treating this problem.

The basic equations are continuity, Navier-Stokes, energy and mass diffusion resulting from various conservation laws.

These equations, in general form are [14]

continuity:

$$\frac{\partial \rho}{\partial t} + \vec{\nabla} \cdot (\rho \vec{V}) = 0 \tag{2.1}$$

momentum:

$$\rho \frac{\overrightarrow{DV}}{\overrightarrow{Dt}} = \rho \left[\frac{\partial \vec{V}}{\partial t} + (\vec{\nabla} \cdot \vec{\nabla}) \vec{V} \right]$$

$$= \rho \vec{g} - \vec{\nabla} p + \mu \nabla^2 \vec{V} + \frac{\mu}{3} \vec{\nabla} (\vec{\nabla} \cdot \vec{V}) \qquad (2.2)$$

energy:

$$\rho \frac{\text{De}}{\text{Dt}} = \rho \left[\frac{\partial e}{\partial \mathbf{t}} + (\vec{\mathbf{V}} \cdot \vec{\nabla}) e \right]
= \vec{\nabla} \cdot (\vec{\mathbf{K}} \cdot \vec{\nabla} \cdot \vec{\mathbf{T}}) + q^{m} - p \vec{\nabla} \cdot \vec{\mathbf{V}} + \mu \varphi$$
(2.3)

species:

$$\frac{\partial C}{\partial t} + (\vec{V} \cdot \vec{\nabla}) C = \vec{\nabla} \cdot (\vec{D} \vec{\nabla} c) + C'''$$
(2.4)

where \vec{V} is the fluid velocity vector. T is the temperature, e is the specific internal energy of the mixture, c is the concentration of a single diffusing species defined as the ratio of mass of the species (in a given volume) to the mass of mixture in the same volume, q''' and c''' are the rates of energy and species generation respectively per unit volume, \vec{g} is the gravitational force per unit volume, p is the pressure, μ , K, D are the molecular transport properties namely dynamic viscosity, conductivity, and mass diffusivity and ϕ is the function associated with the dissipation of energy.

2.2. Approximations:

Considering the fluid to behave as a perfect gas, we can rewrite the energy equation (2.3) as

$$\rho C_{p} \stackrel{DT}{\overline{Dt}} = \vec{\nabla} \cdot (\vec{K} \vec{\nabla} T) + q''' + \frac{Dp}{Dt} + \mu \phi \qquad (2*5)$$

where e_{p} is the specific heat at constant pressure for the mixture.

The principal difficulties in the above equations (2.1), (2.2), (2.4) and (2.5) result mainly from the possible variation of transport properties μ , K and D on the one hand and density ρ on the other hand. Since μ , K and D are dependent primarily on temperature, an appreciable variation occurs only in those processes involving large temperature differences. Hence these properties are assumed constant here. Their variation can however be easily accounted for in the numerical method.

The density differences are approximated, for processes not involving large temperature differences, by the Boussinesq approximation [15]. This simplification renders the continuity equation to the constant density form, and introduces into the momentum equation a buoyancy force arising from both temperature and concentration differences.

For natural convection flows, from the hydrostatic considerations, the pressure gradient $\vec{\nabla}P$ in the remote ambient fluid is ρ_{∞} \vec{g} where ρ_{∞} is the density of ambient fluid.

$$\therefore \rho \vec{g} - \vec{\nabla} P = \vec{g} (\rho - \rho_{\infty}) .$$

Now since the surface is vertical and co-ordinate x is assumed positive upwards, as shown in fig. 2.1 the only term of the body force is $-\rho g_x$ and $\frac{\partial P}{\partial x} = -\rho_x g_x$

^{*}variation with concentration neglected due to low mass fraction of diffusing species.

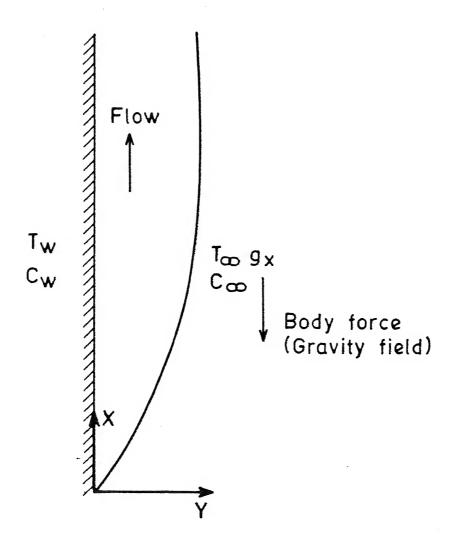


Fig. 2.1 Coordinate system for combined force and forced convection flow over a vertical flat plate.

$$\rho \vec{g} - \vec{\nabla} P = g_{x} (\rho_{\infty} - \rho)$$
 (2.6a)

The series expansion of $(\rho_{\infty} - \rho)$ in terms of T, p and c, at a given location can be written as

$$(\rho_{\infty} - \rho) = \rho_{\beta} (T - T_{\infty}) + \rho_{\beta}^{*} (c - c_{\infty})$$
 (2.6b)

where β is the volumetric coefficient of thermal expansion, β^* is the volumetric coefficient of expansion with concentration, ${
m T}_{\infty}$ and ${
m c}_{\infty}$ are the temperature and concentration respectively in the free stream.

The pressure term $\frac{DP}{Dt}$ in the energy equation (2.5) is negligible for gas flows of small vertical extent as is the case here. The viscous dissipation term is also negligible for small velocity flows. Moreover the rate of energy generation q"' and the rate of species generation c'" due to chemical reaction are assumed to be zero.

2.3. Governing Equations for Natural Convection:

With all approximations and assumptions discussed above the equations become

$$\vec{\nabla} \cdot \vec{V} = 0 \tag{2.7}$$

$$\rho \left[\frac{\partial \vec{V}}{\partial t} + (\vec{V} \cdot \vec{\nabla}) \vec{V} \right] = \rho g_X \beta (T - T_{\infty}) + \rho g_X \beta (c - c_{\infty}) + \mu \nabla^2 \vec{V}$$
(2.8)

$$\rho C_{p} \left[\frac{\partial T}{\partial t} + (\vec{V} \cdot \vec{\nabla}) T \right] = K \nabla^{2} T$$
 (2.9)

$$\frac{\partial C}{\partial t} + (\vec{V} \cdot \vec{\nabla}) C = D \vec{\nabla} C \qquad (2.10)$$

The present study is concerned with the simple case of two dimensional flow where $\vec{V} = (u,v)$. In terms of the co-ordinate system shown in fig. ?.1, the equations (2.7) to (2.10) can be written as,

continuity:

$$\frac{\partial \mathbf{u}}{\partial \mathbf{x}} + \frac{\partial \mathbf{v}}{\partial \mathbf{y}} = 0 \tag{2.11}$$

x-momentum:

$$\rho(\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y}) = \rho g\beta(T - T_{\infty}) + \rho g\beta*(c - c_{\infty}) + \mu(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2})$$
(2.12a)

y-momentum:

$$\rho(\frac{\partial \mathbf{v}}{\partial t} + \mathbf{u} \frac{\partial \mathbf{v}}{\partial \mathbf{x}} + \mathbf{v} \frac{\partial \mathbf{v}}{\partial \mathbf{y}}) = \mu(\frac{\partial^2 \mathbf{v}}{\partial \mathbf{x}^2} + \frac{\partial^2 \mathbf{v}}{\partial \mathbf{y}^2})$$
 (2.12b)

energy:

$$\rho C_{p} \left(\frac{\partial T}{\partial t} + u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} \right) = K \left(\frac{\partial^{2} T}{\partial x^{2}} + \frac{\partial^{2} T}{\partial y^{2}} \right)$$
 (2.13)

species:

$$\frac{\partial c}{\partial t} + u \frac{\partial c}{\partial x} + v \frac{\partial c}{\partial y} = D(\frac{\partial^2 c}{\partial x^2} + \frac{\partial^2 c}{\partial y^2})$$
 (2.14)

These equations can be further simplified by carrying out an order of magnitude analysis [15]. Let us first non-dimensionalize the variables as

$$\vec{u} = \vec{u}$$
, $\vec{v} = \vec{v}$, $\vec{x} = \vec{x}$, $\vec{y} = \vec{y}$, $\theta = \frac{T - T_{\infty}}{T_{\omega} - T_{\infty}}$, $C = \frac{C - C_{\infty}}{C_{\omega} - C_{\infty}}$

where L, U are the characteristic length and velocity, T_W and c_W are the wall temperature and concentration. Equations (2.11) to (2.14) in dimensionless form are

(2•18)

$$\frac{\partial \overline{u}}{\partial \overline{x}} + \frac{\partial \overline{v}}{\partial \overline{y}} = 0$$

$$o(1) \quad o(1)$$

$$\frac{\partial \overline{u}}{\partial \overline{t}} + \overline{u} \frac{\partial \overline{u}}{\partial \overline{x}} + \overline{v} \frac{\partial \overline{u}}{\partial \overline{y}} = \theta + NC + \frac{1}{\sqrt{Gr}} (\frac{\partial^2 \overline{u}}{\partial \overline{x}^2} + \frac{\partial^2 \overline{u}}{\partial \overline{y}^2})$$

$$o(1) \quad o(1) \quad o(1) \quad o(\delta) \quad o(\frac{1}{\delta}) \quad o(1) \quad o(1) \quad o(1) \quad o(\frac{1}{\delta^2})$$

$$\frac{\partial \overline{v}}{\partial \overline{t}} + \overline{u} \frac{\partial \overline{v}}{\partial \overline{x}} + \overline{v} \frac{\partial \overline{v}}{\partial \overline{y}} = \frac{1}{\sqrt{Gr}} (\frac{\partial^2 \overline{v}}{\partial \overline{x}^2} + \frac{\partial^2 \overline{v}}{\partial \overline{y}^2}) \qquad (2.16b)$$

$$o(\delta) \quad o(1) \quad o(\delta) \quad o(\delta) \quad o(1) \quad o(\delta) \quad o(\frac{1}{\delta})$$

$$\frac{\partial \theta}{\partial \overline{t}} + \overline{u} \frac{\partial \theta}{\partial \overline{x}} + \overline{v} \frac{\partial \theta}{\partial \overline{y}} = \frac{1}{\Pr \sqrt{Gr}} (\frac{\partial^2 e}{\partial \overline{x}^2} + \frac{\partial^2 e}{\partial \overline{y}^2}) \qquad (2.17)$$

$$o(1) \quad o(1) \quad o(1) \quad o(\delta) \quad o(\frac{1}{\delta_t}) \qquad o(1) \quad o(\frac{1}{\delta_t^2})$$

$$\frac{\partial C}{\partial \overline{t}} + \overline{u} \frac{\partial C}{\partial \overline{y}} + \overline{v} \frac{\partial C}{\partial \overline{y}} = \frac{1}{\Pr \sqrt{Gr}} (\frac{\partial^2 c}{\partial \overline{x}^2} + \frac{\partial^2 c}{\partial \overline{y}^2}) \qquad (2.18)$$

where N = $[\beta^*(c_w-c_\infty)]/[\beta(T_w-T_\infty)]$ measures the relative importance of chemical and thermal effects in causing the density differences which create the natural convection effect. Gr = g β L³ (T_w-T_w)/ ν^2 is the thermal Grashof number, $Gr^* = g\beta^*L^3 (c_W^- - c_\infty)/\nu^2$ is the mass Grashof number, $Pr = \frac{\mu c_p}{\kappa}$ is the Prandtl number, and $Sc = \frac{D}{\alpha}$ is the Schmidt number, and where the order of magnitude of each term is written under it according to the following estimation.

 $o(1) \ o(1)o(1) \ o(\delta)o(\frac{1}{\delta_0})$

 $o(1) \quad o(\frac{1}{\epsilon^2})$

In the continuity equation $\frac{\partial u}{\partial x}$ must be of order 1 since both \bar{u} and \bar{x} are of order 1. Therefore $\frac{\partial v}{\partial v}$ is also of order 1. Since \bar{y} is of order δ for the hydrodynamic boundary layer, \bar{v} is also of order δ . Clearly δ is the hydrodynamic boundary layer thickness. In the x-direction momentum equation (2.16a), we can therefore neglect $\frac{\partial^2 \bar{u}}{\partial \bar{x}^2} = O(1)$ as compared to $\frac{\partial^2 \bar{u}}{\partial \bar{y}^2} = O(\frac{1}{\delta^2})$. Also, viscous forces must be of the same order as the inertia forces and as the buoyancy forces. Therefore $\delta = O(Gr^{-1/4})$. Considering the y-momentum equation (2.16b) it is observed that the inertia and viscous terms are of order δ or less. Hence as compared to the x-momentum equation the whole of y-momentum equation can be neglected.

For conduction and convection terms in the energy equation (2.17) to be comparable δ_t = thermal boundary layer thickness = 0 (Pr^{-1/2} Gr^{-1/4}). Similarly an estimate of the concentration boundary layer thickness δ_c is δ_c = 0(Sc^{-1/2}Gr^{-1/4}). Also the terms $\frac{\partial^2 \theta}{\partial \overline{x}^2}$ and $\frac{\partial^2 c}{\partial \overline{x}^2}$ are negligible as compared to $\frac{\partial^2 \theta}{\partial \overline{y}^2}$ and $\frac{\partial^2 c}{\partial \overline{y}^2}$ respectively, and can therefore be neglected.

This analysis of relative order of magnitude yields the following system of boundary layer-type equations governing the distributions of u,v,T and c for transient free convection over a vertical flat plate.

continuity:

$$\frac{\partial \mathbf{u}}{\partial \mathbf{x}} + \frac{\partial \mathbf{v}}{\partial \mathbf{y}} = 0 \tag{2.19}$$

momentum:

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = y \frac{\partial^2 u}{\partial y^2} + \beta g(T - T_{\infty}) + \beta g(c - c_{\infty})$$
(2.20)

energy:

$$\frac{\partial T}{\partial t} + u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} = \alpha \frac{\partial^2 T}{\partial y^2}$$
 (2.27)

species:

$$\frac{\partial c}{\partial t} + u \frac{\partial c}{\partial x} + v \frac{\partial c}{\partial y} = D \frac{\partial^2 c}{\partial y^2}$$
 (2.22)

where we have returned to dimensional variables u,v,T and c and where ν is the kinematic viscosity (μ/ρ) and α is the thermal diffusivity $(K/\rho C_n)$.

2.4 Boundary Conditions And Initial Conditions:

For an isothermal plate, the boundary and initial conditions are

$$u(x,y,0) = 0; T(x,y,0) = T_{\infty}; c(x,y,0) = c_{\infty}$$

$$u(x,0,t) = 0; T(x,0,t) = T_{W}; c(x,0,t) = c_{W}$$

$$u(0,y,t) = u_{\infty}; T(0,y,t) = T_{\infty}; c(0,y,t) = c_{\infty}$$

$$u(x,\infty,t) = u_{\infty}; T(x,\infty,t) = T_{\infty}; c(x,\infty,t) = c_{\infty}$$

$$v(x,y,0) = 0;$$

$$v(x,0,t) = 0;$$

where u_{∞} is the free stream velocity in the same direction as that induced by free convection i.e. aiding flow. When $u_{\infty}=0$ the problem changes to that of pure natural convection.

2.5. Non-dimensionalisation:

By the following choice of dimensionless parameters suggested by the above order of magnitude analysis

$$X = \frac{X}{L} \qquad ; \qquad Y = \frac{Y}{L \text{ Gr}^{1/4}} \qquad ; \qquad \tau = \frac{t \quad v \text{ Gr}^{1/2}}{L^2}$$

$$U = \frac{uL}{v \text{ Gr}^{1/2}} ; \qquad V = \frac{v \text{ L}}{v \text{ Gr}^{1/4}}$$

$$\theta = \frac{T - T_{\infty}}{T_{w} - T_{\infty}} \qquad ; \qquad C = \frac{c - c_{\infty}}{c_{w} - c_{\infty}}$$

equations (2.19) through (2.22) and initial and boundary conditions are expressed in dimensionless form as

$$\frac{\partial U}{\partial X} + \frac{\partial V}{\partial Y} = 0 \tag{2.24}$$

$$\frac{\partial U}{\partial \tau} + U \frac{\partial U}{\partial x} + V \frac{\partial U}{\partial y} = \frac{\partial^2 U}{\partial y^2} + \theta + NC$$
 (2.25)

$$\frac{\partial \theta}{\partial \tau} + U \frac{\partial \theta}{\partial X} + V \frac{\partial \theta}{\partial Y} = \frac{1}{Pr} \frac{\partial^2 \theta}{\partial Y^2}$$
 (2.26)

$$\frac{\partial C}{\partial \tau} + U \frac{\partial C}{\partial X} + V \frac{\partial C}{\partial Y} = \frac{1}{Sc} \frac{\partial^2 C}{\partial Y^2}$$
 (2.27)

$$U(X,Y,0) = 0 \quad ; \quad \theta(X,Y,0) = 0 \quad ; \quad C(X,Y,0) = 0$$

$$U(X,0,\tau) = 0 \quad ; \quad \theta(X,0,\tau) = 1 \quad ; \quad C(X,0,\tau) = 1$$

$$U(0,Y,\tau) = U_{\infty} \quad ; \quad \theta(0,Y,\tau) = 0 \quad ; \quad C(0,Y,\tau) = 0$$

$$U(X,\infty,\tau) = U_{\infty} \quad ; \quad \theta(X,\infty,\tau) = 0 \quad ; \quad C(X,\infty,\tau) = 0$$

$$V(X,Y,0) = 0$$

$$V(X,0,\tau) = 0$$

where $U_{\infty} = \frac{U_{\infty} L}{\nu Gr^{1/2}} = \text{Re } Gr^{-1/2}$.

Equations (2.24) through (2.27) and (2.28) show that dependent variables U, V, θ and C are functions of the dimensionless spatial coordinates X and Y, dimensionless time τ , and dimensionless parameters N, Pr, Sc and U... Note that for forced flow, $\tau = (t \ u_{\infty})/(LU)$ which is a proper dimensionless time.

N measures the relative importance of chemical and thermal diffusion in causing the density differences, which create the natural convection effect. N is equal to zero when there is no species diffusion body force and becomes infinite for no thermal diffusion. The momentum equation (2.25) indicates that N is positive or negative according as the mass diffusion forces aid or oppose those of thermal diffusion. $U_{\infty} = \text{Re Gr}^{-1/2}$ is the parameter which determines the extent of contribution of forced convection to natural convection. $U_{\infty} = 0$ is the case of pure natural convection while $U_{\infty} > 0$ is aiding flow and $U_{\infty} < 0$ is the opposing flow. Negative values of U_{∞} can cause separation [16] which can not be handled by the boundary layer type equations (2.24) to (2.27).

In the case of pure forced convection, the Prandtl number $(Pr = \nu/\alpha)$ relates the relative thicknesses of the momentum and thermal boundary layers δ and δ_{t} respectively. Similarly the Schmidt number $(Sc = \nu/D)$ in the pure forced convective mass transfer, relates the momentum and concentration boundary layer thicknesses for δ and δ_{C} .

However, in the case of free convection or combined free and forced convection, in the presence of mass diffusion contribution to the buoyancy force, the relationship amongst δ , $\delta_{\rm t}$ and $\delta_{\rm c}$ is extremely complex and depends upon Sc, Pr, U and the buoyancy ratio parameter N.

^{*} For N $\rightarrow \infty$, Gr should be replaced by Gr* for non-dimensionalization

2.6. Heat and Mass Transfer Analysis:

It is a common practice to express heat transfer and mass transfer characteristics in terms of the flux rate divided by the temperature or concentration difference, causing the heat and mass transfer respectively. This ratio defines the heat and mass-transfer coefficients h and h_D respectively, with the help of which we can define the instantaneous local Nusselt and Sherwood numbers as

$$Nu_{x} = \frac{hx}{K} = \frac{q x}{K(T_{w} - T_{\infty})}$$
 (2.29a)

$$Sh_{x} = \frac{h_{D}^{x}}{D} = \frac{\tilde{m} x}{D(c_{w} - c_{\infty})}$$
 (2.29b)

where \hat{q} and \hat{m} are the heat and mass flux rates respectively. For uniform values of temperature and concentration differences $(T_w^-T_\infty)$ and $(c_w^-c_\infty)$, local values, h and h_D , and average values \hat{h} and \hat{h}_D are of interest.

These values are expressed in the dimensionless form to obtain the instantaneous local Nusselt and Sherwood numbers respectively as follows

$$Nu_{x} = -\left(\frac{\partial \theta}{\partial Y}\right) \Big|_{Y=0} \times Gr^{1/4}$$
 (2.30a)

$$Sh_{x} = -\left(\frac{\partial C}{\partial Y}\right)\big|_{Y=0} \times Gr^{1/4}$$
 (2.30b)

Since ${\rm Gr}^{1/4}$ is a constant, let us merge it into Nu and Sh so as to redefine the instantaneous local Nusselt and Sherwood numbers as

$$Nu_{x} = -X \left(\frac{\partial \theta}{\partial Y}\right) \Big|_{Y=0}$$
 (2.31a)

$$Sh_{x} = -x \left(\frac{\partial C}{\partial Y} \right) \Big|_{Y=0}$$
 (2.31b)

The instantaneous mean Nusselt number is defined as

$$Nu_{m} = (\frac{\bar{h}L}{\bar{K}}) \quad Gr^{-1/4}$$
where $\bar{h} = \frac{1}{L} \int_{0}^{L} h \, dx = \int_{0}^{1} h \, dx$ (2.32)

This yields

$$Nu_{m} = \int_{0}^{1} \left(-\frac{\partial \theta}{\partial Y}\right) \Big|_{Y=0} dX \qquad (2.33a)$$

Similarly the instantaneous mean Sherwood number can be found from

$$\operatorname{Sh}_{\mathfrak{M}} = \int_{0}^{1} -\left(\frac{\partial C}{\partial Y}\right) \Big|_{Y=0} dX \qquad (2.33b)$$

Chapter 3

FINITE DIFFERENCE FORMULATION

Solutions of the coupled continuity, momentum, energy and species equations (2.24) - (2.27) subjected to initial and boundary conditions (2.28), were obtained using a numerical marching procedure. Numerical marching procedures are methods in which the solution is obtained in a step-by-step manner, always moving downstream through the flow field, and forward in time. A number of finite difference forms for the representation of the above equations are possible. These are the implicit, Crank-Nicholson form etc.

The choice of finite difference representation depends on many factors, including the problem itself, and the size and speed of computation desired. Explicit difference representations are those in which the unknown quantities in the equation may be solved for one at a time, as each step in the marching direction is taken. While simple to work with, explicit methods are prone to instability and require impracticably small step sizes in order to ensure stability. Implicit representations require the solution of a set of simultaneous equations for the unknowns as each step is taken in the downstream direction or in time. They have no stability constraints and thus allow large steps to be taken without any problem.

However, almost all formulations for transient problems, appearing in the literature are explicit. Callhan and Marner [13] too, have solved the present problem using explicit representation. We use the implicit formulation nevertheless in order to take advantage of its universal stability so long as $U \ge O_{\bullet}$ This means in particular that there is no restriction on the size of steps AT or AX. In using the explicit formulation, such small values of $\Delta \tau$ and ΔX are required that computations can become extremely time consuming. Since the matrices encountered in the implicit formulation discussed here are tridiagonal, the computational time required for a complete set of calculations at each step is approximately the same as that required for the explicit method, but much larger values of $\Delta \tau$ and ΔX are permitted by the implicit method. Thus the implicit method seems to be superior to the explicit method.

3.1 Finite-Difference Equations:

A non-uniform finite difference grid (Fig. 3.1) is imposed on the flow field. Step sizes ΔX , ΔY and ΔT are taken respectively in the X and Y direction and in time. Subscripts j+1 and k indicate the location of the point under consideration in x and y directions respectively, while the subscript (i+1) indicates the current time.

The difference form selected for equations (2.25), (2.26) and (2.27) is highly implicit in that not only are all

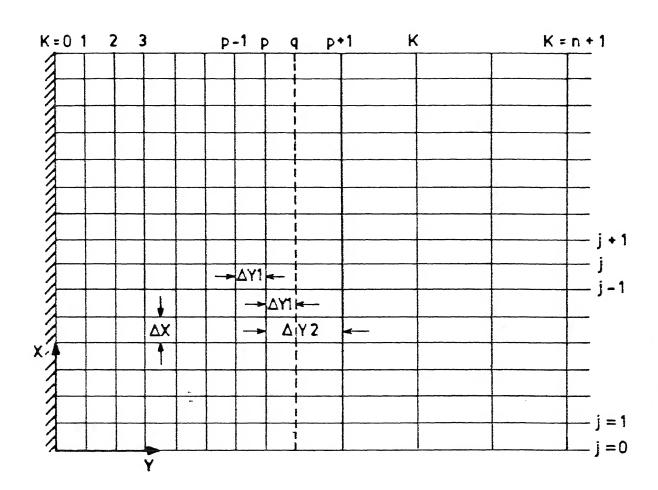


Fig. 3.1 Finite Difference grid with variable mesh size.

Y-derivatives evaluated at j+1 but the coefficients of nonlinear convective terms are also evaluated at j+1. This is essential for free convection flow since if the usual implicit form is chosen it results in U velocity profile decreasing linearly from the plate to zero at whatever value of Y is chosen as infinity. This result is obviously incorrect. Hence the highly implicit scheme is used.

The finite difference forms chosen for equations (2.24) to (2.27) are

$$\frac{U_{j+1,k+1,i+1}-U_{j,k+1,i+1}}{\Delta x} + \frac{V_{j+1,k+1,i+1}-V_{j+1,k,i+1}}{\Delta y} = 0 (3.1)$$

$$\frac{U_{j+1,k,i+1}-U_{j+1,k,i}}{\Delta \tau}+U_{j+1,k,i+1}$$

$$\frac{U_{j+1,k,i+1}-U_{j,k,i+1}}{\Delta X}$$

+
$$V_{j+1,k,i+1}$$
 $U_{j+1,k+1,i+1} - U_{j+1,k-1,i+1}$ $U_{j+1,k-1,i+1} - U_{j+1,k-1,i+1}$

$$= \frac{U_{j+1,k+1,i+1} - 2U_{j+1,k,i+1} + U_{j+1,k-1,i+1}}{(\Delta Y)^{2}} + \theta_{j+1,k,i+1}$$

+
$$N C_{j+1,k,i+1}$$
 (3.2)

$$\frac{\theta_{j+1,k,i+1} - \theta_{j+1,k,i}}{\Delta \tau} + U_{j+1,k,i+1} \frac{\theta_{j+1,k,i+1} - \theta_{j,k,i+1}}{\Delta x} + V_{j+1,k,i+1} \frac{\theta_{j+1,k,i+1} - \theta_{j,k,i+1}}{2(\Delta Y)}$$

$$= \frac{1}{\Pr} \frac{\theta_{j+1,k+1,i+1} - 2\theta_{j+1,k,i+1} + \theta_{j+1,k-1,i+1}}{(\Delta Y)^2}$$
 (3.3)

$$\frac{C_{j+1,k,i+1}-C_{j+1,k,i}}{\Delta \tau} + U_{j+1,k,i+1} \frac{C_{j+1,k,i+1}-C_{j,k,i+1}}{\Delta X}$$

+
$$V_{j+1,k,i+1}$$
 $\frac{C_{j+1,k+1,i+1} - C_{j+1,k-1,i+1}}{2(\Delta Y)}$

$$= \frac{1}{Sc} \frac{C_{j+1,k+1,i+1} - 2C_{j+1,k,i+1} + C_{j+1,k-1,i+1}}{(\Delta Y)^2}$$
 (3.4)

Truncation error is of $o(\Delta X)$ and $o(\Delta Y)^2$ for momentum, energy and concentration equations, and of $o(\Delta X)$ and $o(\Delta Y)$ for the continuity equation. Since the differential formulation given above is non-linear, none of the techniques for linear algebraic equations may be employed. However, one very simple and effective iterative technique is used here.

To start with equations (3.1) to (3.4) are rewritten using superscripts to indicate on which iteration that value was obtained, for example $U_{j+1,k,i+1}^{(1)}$ is obtained on the (1)th iteration while $U_{j+1,k,i+1}^{(1+1)}$ is obtained on the (1+1)th iteration. In such a linearised form equations (3.1) to (3.4) are rewritten as

$$\frac{V_{j+1,k,i+1}^{(1+1)} - V_{j,k,i+1}^{(1+1)} - V_{j+1,k+1,i+1}^{(1+1)} - V_{j+1,k,i+1}^{(1+1)}}{\Delta x} + \frac{V_{j+1,k+1,i+1}^{(1+1)} - V_{j+1,k,i+1}^{(1+1)}}{\Delta y} = 0$$
 (3.5)

$$\frac{U_{j+1,k,i+1}^{(1+1)} - U_{j+1,k,i}}{\Delta \tau} + U_{j+1,k,i+1}^{(1)} + U_{j+1,k,i+1}^{(1+1)} - U_{j,k,i+1}^{(1+1)} - U_{j,k,i+1}^{(1+1)} - U_{j+1,k,i+1}^{(1+1)} - U_{j+1,k-1,i+1}^{(1+1)} \\
+ V_{j+1,k,i+1}^{(1)} - 2U_{j+1,k,i+1}^{(1+1)} + U_{j+1,k-1,i+1}^{(1+1)} \\
= \frac{U_{j+1,k+1,i+1}^{(1+1)} - 2U_{j+1,k,i+1}^{(1+1)} + U_{j+1,k-1,i+1}^{(1+1)}}{(\Delta Y)^{2}} \\
+ \theta_{j+1,k,i+1}^{(1)} - \theta_{j+1,k,i}^{(1)} + U_{j+1,k,i+1}^{(1+1)} - \theta_{j+1,k,i+1}^{(1+1)} - \theta_{j,k,i+1}^{(1+1)} - \theta_{j+1,k-1,i+1}^{(1+1)} \\
+ V_{j+1,k,i+1}^{(1+1)} - \frac{\theta_{j+1,k+1,i+1}^{(1+1)} - \theta_{j+1,k-1,i+1}^{(1+1)}}{2(\Delta Y)} \\
= \frac{1}{Pr} \frac{\theta_{j+1,k+1,i+1}^{(1+1)} - 2\theta_{j+1,k,i+1}^{(1+1)} + \theta_{j+1,k-1,i+1}^{(1+1)}}{(\Delta Y)^{2}}$$

$$\frac{C_{j+1,k,i+1}^{(1+1)} - C_{j+1,k,i}}{\Delta \tau} + U_{j+1,k,i+1}^{(1+1)} - \frac{C_{j,k,i+1}^{(1+1)} - C_{j,k,i+1}^{(1+1)}}{\Delta X}$$

$$\frac{\Delta \tau}{\Delta \tau} + \frac{U_{j+1,k,i+1}^{(1+1)}}{\Delta x} + \frac{U_{j+1,k,i+1}^{(1+1)}}{\Delta x} + \frac{C_{j+1,k+1,i+1}^{(1+1)}}{\Delta x} + \frac{C_{j+1,k+1,i+1}^{(1+1)}}{2(\Delta y)} + \frac{C_{j+1,k+1,i+1}^{(1+1)}}{2(\Delta y)} + \frac{C_{j+1,k+1,i+1}^{(1+1)}}{2(\Delta y)^{2}} + \frac{C_{j+1,k+1,i+1}^{(1+1)}}{2(\Delta y)^{2}}$$

$$= \frac{1}{Sc} \frac{C_{j+1,k+1,i+1}^{(1+1)} - 2C_{j+1,k,i+1}^{(1+1)} + C_{j+1,k-1,i+1}^{(1+1)}}{(\Delta y)^{2}}$$
(3.8)

To continuity equation (3.5) and conservation equations (3.6) to (3.8) for momentum, energy and chemical species respectively are written in more useful form as

$$\begin{split} v_{j+1,k+1,i+1}^{(1+1)} &= v_{j+1,k,i+1}^{(1+1)} - \frac{\Delta Y}{\Delta X} (v_{j+1,k+1,i+1}^{(1+1)} - v_{j,k+1,i+1}^{(1)}) \\ &= \frac{1}{(\Delta Y)^2} - \frac{v_{j+1,k,i+1}^{(1)}}{2(\Delta Y)} v_{j+1,k-1,i+1}^{(1+1)} + \frac{1}{\Delta \tau} + \frac{2}{(\Delta Y)^2} + \frac{v_{j+1,k,i+1}^{(1)}}{\Delta X} \end{split}$$

$$(3.9)$$

$$\begin{bmatrix} \frac{-1}{(\Delta Y)^2} - \frac{v_{j+1,k,i+1}^{(1)}}{2(\Delta Y)} v_{j+1,k,i+1}^{(1+1)} + \frac{1}{\Delta \tau} + \frac{2}{(\Delta Y)^2} + \frac{v_{j+1,k,i+1}^{(1)}}{\Delta X} \end{bmatrix}$$

$$= \frac{v_{j+1,k,i+1}^{(1+1)} + v_{j+1,k,i+1}^{(1)}}{2(\Delta Y)} v_{j+1,k,i+1}^{(1)} + v_{j+1,k,i+1}^{(1)} + v_{j+1,k,i+1}^{(1)} + v_{j+1,k,i+1}^{(1)} \end{bmatrix}$$

$$= \frac{v_{j+1,k,i+1}^{(1+1)} + v_{j+1,k,i+1}^{(1+1)}}{2(\Delta Y)} v_{j+1,k,i+1}^{(1+1)} + v_{j+1,k,i+1}^{(1+1)}}{2(\Delta Y)} v_{j+1,k+1,i+1}^{(1+1)} + v_{j+1,k,i+1}^{(1+1)} v_{j+1,k,i+1}^{(1+1)} \end{bmatrix}$$

$$= \frac{v_{j+1,k,i}^{(1+1)} + v_{j+1,k,i+1}^{(1+1)}}{\Delta X} v_{j+1,k,i+1}^{(1+1)} v_{j+1,k,i+1}^{(1+1)} v_{j+1,k,i+1}^{(1+1)} v_{j+1,k,i+1}^{(1+1)}}{\Delta X} v_{j+1,k,i+1}^{(1+1)} v$$

$$+ \frac{v_{j+1,k,i+1}^{(1+1)}}{\Delta x} \right] c_{j+1,k,i+1}^{(1+1)} + \left[\frac{-1}{sc(\Delta Y)^{2}} + \frac{v_{j+1,k,i+1}^{(1+1)}}{2(\Delta Y)} \right] c_{j+1,k+1,i+1}^{(1+1)}$$

$$= \frac{C_{j+1,k,i}}{\Delta \tau} + \frac{U_{j+1,k,i+1}^{(1+1)} C_{j,k,i+1}}{\Delta X}$$
 (3.12)

Equations (3.10) to (3.12) written for k = 1(1)n constitute sets of n linear algebraic equations in n unknowns $U_{j+1,k,j+1}^{(1+1)}$, $U_{j+1,k,j+1}^{(1+1)}$ and $U_{j+1,k,j+1}^{(1+1)}$ respectively. Value of n is chosen large enough, so that on several points of the grid close to k = n the U velocities are essentially that of free stream. Solution procedure and steps followed for the solution of the finite difference equations (3.9) to (3.12) to be explained later.

3.2 Heat and Mass Transfer Solution:

In finite difference form equations (2.31a) and (2.31b) for the instantaneous local Nusselt and Sherwood numbers can be written as

$$Nu_{x} = X = \frac{3\theta_{j+1,0} - 4\theta_{j+1,1} + \theta_{j+1,2}}{2(\Delta Y)}$$
 (3.13a)

$$Sh_{x} = X = \frac{3C_{j+1,0} - 4C_{j+1,1} + C_{j+1,2}}{2(\Delta Y)}$$
 (3.13b)

where three-point forward differences have been used to evaluate $(\frac{\partial \, \theta}{\partial \, Y})$ and $(\frac{\partial \, C}{\partial \, Y})$ at the plate. These differences involve an error of $o(\Delta Y)^2$.

For convenience in expressing the instantaneous mean Nusselt and Sherwood numbers, let \$ and \$ represent the following values at any instant

$$\xi = -(\frac{\partial \theta}{\partial Y}) \Big|_{Y=0}$$
 and $\zeta = -(\frac{\partial C}{\partial Y}) \Big|_{Y=0}$ (3.14)

Then the instantaneous mean Nusselt and Sherwood numbers can be written as (cf. equations (2.33a) and (2.33b))

$$Nu_{m} = \int_{0}^{1} \xi dx \qquad (3.15a)$$

$$Sh_{m} = \int_{0}^{1} \zeta dX \qquad (3.15b)$$

Using Simpson's rule [17], these can be written as

$$Nu_{m} = \frac{\Delta X}{3} \left[\xi_{0} + 4\xi_{1} + 2\xi_{2} + 4\xi_{3} + \cdots + 2\xi_{m-2} + 4\xi_{m-1} + \xi_{m} \right]$$
 (3.16a)

$$Sh_{m} = \frac{\Delta X}{3} \left[\zeta_{0} + 4\zeta_{1} + 2\zeta_{2} + 4\zeta_{3} + \cdots + 2\zeta_{m-2} + 4\zeta_{m-1} + \zeta_{m} \right]$$
 (3.16b)

where m must be an even number.

3.3. Computational Steps:

For the solution of finite-difference equations in Section 3.1 and Section 3.2 following procedure is followed.

Starting from the specified initial conditions at time $\tau=0$, we marched in time first and the velocity, temperature and concentration fields, and mean Nusselt and Sherwood numbers

were obtained at time $\tau + \Delta \tau$, for each X location, starting from the leading edge and marching downstream. At a particular location X and time τ , the iterative technique is applied as explained below.

- The first iteration is started by guessing values for $U^{(0)}_{j+1,k,i+1}, V^{(0)}_{j+1,k,i+1}, V^{(0)}_{j+1,k,i+1}, V^{(0)}_{j+1,k,i+1}, V^{(0)}_{j+1,k,i+1}, V^{(0)}_{j+1,k,i+1}$ These guesses are the values at the preceding step upstream (like $U_{j,k,i+1}$ etc.)
- b) Taking 1 = 0 in equation (3.10), U⁽¹⁾_{j+1,k,i+1} are calculated solving a set of (n) simultaneous equations. (solution procedure to be explained later.)
- c) The continuity equation (3.9) again at 1 = 0 is solved for the transverse velocity component V(1) in the stepwise manner, working outward from the plate surface.
- d) $\theta_{j+1,k,i+1}^{(1)}$ and $C_{j+1,k,i+1}^{(1)}$ are calculated at l=0 from equations (3.11) and (3.12) respectively, solving sets of n simultaneous equations (solution procedure to be explained later.)
- for l = 1,2,3... and so on, until U(1+1) and U(1) y+1,k,i+1 agree to within a desired degree of accuracy s, taken here as 10⁻³. Similar degree of accuracy is set for V_{j+1,k,i+1} end C_{j+1,k,i+1} and C_{j+1,k,i+1}

It might be noted that this iterative procedure is a composite of Jacobi and Gauss-Siedel iterative techniques as extended to nonlinear equations.

Once iteration is complete at a particular location X and time T, the instantaneous local Nusselt and Sherwood numbers are calculated solving equations (3.13a) and (3.13b) respectively.

Now another step ΔX downstream is taken and the process is repreated at the same τ . When the solution has been carried downstream as far as desired, the instantaneous mean Nusselt and Sherwood numbers are calculated solving the equations (3.16a) and (3.16b). Then another time step ΔT is taken, and again starting at the leading edge, the solution is marched spacewise downstream. The whole process is repeated as many times as necessary to determine the steady state solution such that U_{j+1} , k, i and U_{j+1} , k, i+1 agree to within the desired degree of accuracy ϵ .

3.4 Solution Procedure:

At each iteration, the system of (n) simultaneous linear equations resulting from momentum equation (3.10) can be written in matrix form as

Similarly the systems of (n) simultaneous linear equations for energy and species respectively resulting from equations (3.11) and (3.12) respectively are

$$\beta_{1}^{\prime(1)} \alpha_{1}^{\prime(1)} \\ \alpha_{2}^{\prime(1)} \beta_{2}^{\prime(1)} \\ \alpha_{3}^{\prime(1)} \beta_{3}^{\prime(1)} \\ \alpha_{3}^{\prime(1)} \beta_{3}^{\prime(1)} \\ \alpha_{3}^{\prime(1)} \beta_{n-1}^{\prime(1)} \\ \alpha_{n-1}^{\prime(1)} \beta_{n-1}^{\prime(1)} \\ \alpha_{n}^{\prime(1)} \beta_{n}^{\prime(1)} \\ \alpha_{n}^{\prime(1)} \beta_{n}^{\prime($$

where

$$\alpha_{k}^{\prime(1)} = \frac{-1}{\Pr(\Delta Y)^{2}} - \frac{V_{j+1,k,j+1}^{(1+1)}}{2(\Delta Y)}$$

$$\beta_{k}^{\prime(1)} = \frac{1}{\Delta \tau} + \frac{2}{\Pr(\Delta Y)^{2}} + \frac{U_{j+1,k,j+1}^{(1+1)}}{\Delta X}$$

$$\Omega_{k}^{\prime(1)} = \frac{-1}{\Pr(\Delta Y)^{2}} + \frac{V_{j+1,k,j+1}^{(1+1)}}{2(\Delta Y)}$$

$$\varphi_{k}^{\prime(1)} = \frac{\theta_{j+1,k,j}}{\Delta \tau} + \frac{U_{j+1,k,j+1}^{(1+1)}}{\theta_{j,k,j+1}^{(1+1)}}$$

$$\alpha_{K}^{"}(1) = \frac{-1}{\text{Sc}(\Delta Y)^{2}} - \frac{V_{j+1}, K, j+1}{2(\Delta Y)}$$

$$\beta_{K}^{"}(1) = \frac{1}{\Delta \tau} + \frac{2}{\text{Sc}(\Delta Y)^{2}} + \frac{U_{j+1}, K, j+1}{j+1, K, j+1}$$

where

$$\Omega_{k}^{(1)} = \frac{-1}{\operatorname{Sc}(\Delta Y)^{2}} + \frac{V_{j+1,k,i+1}^{(1+1)}}{2(\Delta Y)}$$

$$\varphi_{k}^{(1)} = \frac{C_{j+1,k,i}}{\Delta \tau} + \frac{U_{j+1,k,i+1}^{(1+1)} C_{j,k,i+1}}{\Delta X}.$$

Solution of system (3.17) to (3.19) by the inversion method is highly expensive even on the present-day computers since the number of steps required is of order (n)³ and the storage requirement is of order (n)². As the matrix of coefficients in all the systems (3.17) to (3.19) is tridiagonal a subroutine TRIDIA [16] is used which works efficiently.

3.5. Convergence and Relaxation:

In some cases it is desirable to either overrelax or underrelax the iterative procedure in order to speed up or slow down the changes from iteration to iteration, in the values of dependent variables. Under relaxation is very useful for nonlinear problems. It is used to avoid divergence in the iterative solution of strongly nonlinear problems.

In the present problem after preliminary calculations it was observed that the iterative procedure does not converge unless some underrelaxation is employed. The following relaxation procedure was used.

 employed the values of U's, θ 's and C's are modified as

$$\begin{array}{l} \mathbf{U}_{j+1}^{(1)}, \mathbf{k}, \mathbf{i}+1 < - \mathbf{U}_{j+1}^{(1)}, \mathbf{k}, \mathbf{i}+1 + \lambda_{\mathbf{U}}(\mathbf{U}_{j+1}^{(1+1)}, \mathbf{k}, \mathbf{i}+1 - \mathbf{U}_{j+1}^{(1)}, \mathbf{k}, \mathbf{i}+1) \\ \\ \boldsymbol{\theta}_{j+1}^{(1)}, \mathbf{k}, \mathbf{i}+1 < - \boldsymbol{\theta}_{j+1}^{(1)}, \mathbf{k}, \mathbf{i}+1 + \lambda_{\boldsymbol{\theta}}(\boldsymbol{\theta}_{j+1}^{(1+1)}, \mathbf{k}, \mathbf{i}+1 - \boldsymbol{\theta}_{j+1}^{(1)}, \mathbf{k}, \mathbf{i}+1) \\ \\ \mathbf{C}_{j+1}^{(1)}, \mathbf{k}, \mathbf{i}+1 < - \mathbf{C}_{j+1}^{(1)}, \mathbf{k}, \mathbf{i}+1 + \lambda_{\mathbf{C}}(\mathbf{C}_{j+1}^{(1+1)}, \mathbf{k}, \mathbf{i}+1 - \mathbf{C}_{j+1}^{(1)}, \mathbf{k}, \mathbf{i}+1) \\ \end{array}$$

where λ_U , λ_Θ and λ_C are the underrelaxation factors. The factors found useful in the preliminary calculations were $\lambda_U = \lambda_G = \lambda_C = 0.3$. However as we go downstream they were increased to 1.0. They depend also on the value of U_∞ . Eventhough optimum values were not used effort was made to choose values by trial and error such that computational time was minimized.

3.6. Selection of Step Size:

No quantitative statements can be made about the choice of step sizes. However smaller step sizes are preferred in regions of more rapidly changing velocity, temperature and concentration profiles, in order to reduce the error. Thus a fine mesh size is required close to the plate surface and close to the leading edge. Effects of singularity at the leading edge in the boundary layer problem can be confined to a small region close to the leading edge by selecting small mesh size ΔX until the profiles smooth out somewhat.

Since implicit formulation is used, there is no restriction on the selection of step sizes $\Delta \tau$ and Δx , from the point of view of stability. However, since at small time heat is transferred by conduction only and mass is transferred by diffusion only, and Nu_m and Sh_m are inversely proportional to $\sqrt{\tau}$ in that region, small time-step sizes are used initially and then time-step size is increased to reduce the computational time. Following time-step sizes were used.

for
$$U_{\infty} = 0.0$$
 and 1.0

$$\Delta \tau = 0.05$$
 (0 $\leq \tau \leq 0.4$)

$$\Delta \tau = 0.2$$
 (0.4 $\leq \tau \leq$ steady state τ)

and

for $U_{\infty} = 10.0$

$$\Delta \tau = 0.05 \qquad (0 \le \tau \le 0.2)$$

$$\Delta \tau = 0.2$$
 (0.2 < τ < steady state τ)

In order to reduce the number of equations to be solved, which can effect a considerable saving in computer time, and to keep round-off error accumulated in solving large number of simultaneous equations to a minimum, it is necessary to use variable mesh size. Fine mesh size in regions of rapidly varying velocities and relatively coarse mesh size in regions of slowly varying velocities is called for. Variable mesh sizes employed in Y-direction were as follows:

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$$U_{\infty} = 0.0$$
 and 1.0

$$\Delta \tau = 0.05$$
 (0 $\leq \tau \leq 0.4$)

$$\Delta \tau = 0.2$$
 (0.4 $\leq \tau \leq$ steady state τ)

and

for $U_{\infty} = 10.0$

$$\Delta \tau = 0.05$$
 $(0 \le \tau \le 0.2)$
 $\Delta \tau = 0.2$ $(0.2 \le \tau \le \text{steady state } \tau)$

In order to reduce the number of equations to be solved, which can effect a considerable saving in computer time, and to keep round-off error accumulated in solving large number of simultaneous equations to a minimum, it is necessary to use variable mesh size. Fine mesh size in regions of rapidly varying velocities and relatively coarse mesh size in regions of slowly varying velocities is called for. Variable mesh sizes employed in Y-direction were as follows:

for
$$U_{\infty} = 0.0$$

$$\Delta Y = 0.05 \qquad (0 \le Y \le 0.5)$$

$$\Delta Y = 0.15 \qquad (0.5 \le Y \le 2.0)$$

$$\Delta Y = 0.30 \qquad (2.0 \le Y \le 8.0)$$

$$\Delta Y = 0.50 \qquad (8.0 \le Y \le 17.0)$$
for $U_{\infty} = 1.0$

$$\Delta Y = 0.03 \qquad (0 \le Y \le .30)$$

$$\Delta Y = 0.10 \qquad (.30 \le Y \le 2.0)$$

$$\Delta Y = 0.25 \qquad (2.0 \le Y \le 4.50)$$

$$\Delta Y = 0.50 \qquad (4.50 \le Y \le 11.50)$$
for $U_{\infty} = 10.0$

$$\Delta Y = 0.01 \qquad (0 \le Y \le 0.1)$$

$$\Delta Y = 0.04 \qquad (0.1 \le Y \le 0.5)$$

 $\Delta Y = 0.10 \quad (0.5 \le Y \le 1.5)$

 $\Delta Y = 0.50$ (1.5 $\leq Y \leq 6.0$)

The variable mesh technique provides no difficulty for forward or backward first order differences of error o(h), where h is the mesh size. However, when central differences, either first or second order, are used as in the transverse direction; difficulties arise at the point of mesh size change. To alleviate such a difficulty, consider the mesh size change

from a smaller step size $\Delta Y1$ to a larger step size $\Delta Y2$ at the Y-location k = p in fig. 3.1. Application of the central difference form for the first or second derivative at k = p requires the value at ficticious k = q. This value is found by passing a parabola through the values at k = (p-1), p and (p+1) to yield

$$Q_{j+1,q} = \frac{\Phi_{R}^{-1}}{\Phi_{R}+1} Q_{j+1,p-1} + 2(1-\Phi_{R}) Q_{j+1,p} + 2\frac{\Phi_{R}^{2}}{1+\Phi_{R}} Q_{j+1,p+1}(3-21)$$

where Q is the dependent variable at some instant τ . The derivatives at k = p are then approximated as

$$\frac{2Q}{3Y}\Big|_{k=p} = \frac{Q_{j+1,q} - Q_{j+1,p-1}}{2(\Delta Y_1)}$$
 (3.22a)

$$\frac{\frac{\partial^2 Q}{\partial Y^2}\Big|_{k=p}}{\frac{\partial^2 Q}{\partial Y^2}\Big|_{k=p}} = \frac{\frac{Q_{j+1,q} - 2 Q_{j+1,p} + Q_{j+1,p-1}}{(\Delta Y_1)^2}$$
(3.22b)

where $Q_{j+1,q}$ is given by equation (3.21) and Φ_R is the ratio of small step size to large step size.

At the points of mesh size change equations (3.22a) and (3.22b) are used in the momentum, energy, and species equations and at these points modified equations are rewritten, as they involve central differences in transverse direction.

The difference representation for the continuity equation involves only first order forward differences in the transverse direction. Hence no such modification in equation (3.9) is required. It is ensured that the proper mash size is used in the appropriate region.

Modified equations at points of mesh size change k = p are

Momentum Equation:

$$\frac{U_{j+1,p,i+1}^{(1+1)} - U_{j+1,p,i} + U_{j+1,p,i+1}^{(1)} + U_{j+1,p,i+1}^{(1)} - U_{j,p,i+1}^{(1+1)} - U_{j,p,i+1}^{(1+1)})}{\Delta X} + V_{j+1,p,i+1}^{(1)} + V_{j+1,p,i+1}^{(1)} + \frac{\frac{\delta_{R}^{-1}}{\delta_{R}^{+1}} U_{j+1,p-1,i+1}^{(1+1)} + 2(1-\delta_{R})U_{j+1,p-1,i+1}^{(1+1)} - U_{j+1,p-1,i+1}^{(1+1)}}{2(\Delta Y)} + 2\frac{\frac{\delta_{R}^{-1}}{\delta_{R}^{+1}} U_{j+1,p-1,i+1}^{(1+1)} + 2(1-\delta_{R})U_{j+1,p,i+1}^{(1+1)} + 2\frac{\frac{\delta_{R}^{-1}}{\delta_{R}^{-1}} U_{j+1,p-1,i+1}^{(1+1)} - 2U_{j+1,p,i+1}^{(1+1)} + U_{j+1,p-1,i+1}^{(1+1)}}{(\Delta Y)^{2}} + \theta_{j+1,p,i+1}^{(1)} + N C_{j+1,p,i+1}^{(1)}$$

which is written in the simplified form as

$$\begin{bmatrix} \frac{\Phi_{R}^{-1}}{\Phi_{R}^{+1}} - 1 & V_{j+1,p,i+1}^{(1)} & \frac{\Phi_{R}^{-1}}{\Phi_{R}^{+1}} & U_{j+1,p-1,i+1}^{(1+1)} \\ \frac{2(\Delta Y)}{2(\Delta Y)} & -\frac{(\Delta Y)^{2}}{(\Delta Y)^{2}} \end{bmatrix} U_{j+1,p-1,i+1}^{(1+1)} \\ + \begin{bmatrix} \frac{1}{\Delta T} + \frac{U_{j+1,p,i+1}^{(1)}}{\Delta X} & \frac{2(1-\Phi_{R})}{\Delta X} & V_{j+1,p,i+1}^{(1)} \\ \frac{2(\Delta Y)}{2(\Delta Y)} & +\frac{2\Phi_{R}^{2}}{(\Delta Y)^{2}} \end{bmatrix} U_{j+1,p,i+1}^{(1+1)} \\ + \begin{bmatrix} (\frac{2\Phi_{R}^{2}}{1+\Phi_{R}^{2}}) & (\frac{-1}{(\Delta Y)^{2}} + \frac{V_{j+1,p,i+1}^{(1)}}{2(\Delta Y)}) \end{bmatrix} U_{j+1,p+1,i+1}^{(1+1)} \end{bmatrix}$$

$$= \frac{U_{j+1,p,i}}{\Delta \tau} + \frac{U_{j+1,p,i+1}^{(1)} U_{j,p,i+1}}{\Delta x} + \theta_{j+1,p,i+1}^{(1)} + \theta_{j+1,p,i+1}^{(1)}$$
(3.24)

Similarly energy and concentration equations are written as

$$\begin{split} & \left[\frac{-(1+(\frac{\psi_{R}^{-1}}{\varphi_{R}+1}))}{\Pr(\Delta Y)^{2}} - \frac{(1-(\frac{\psi_{R}^{-1}}{\varphi_{R}+1})) \ V_{j+1,p,j+1}^{(1+1)}}{2(\Delta Y)} \right] \ e_{j+1,p-1,j+1}^{(1+1)} \\ & \left[\frac{1}{\Delta^{\frac{1}{4}}} + \frac{2^{\frac{\phi}{4}}_{R}}{\Pr(\Delta Y)^{2}} + \frac{2^{(1-\phi_{R})} \ V_{j+1,p,j+1}^{(1+1)}}{2(\Delta Y)} + \frac{U_{j+1,p,j+1}^{(1+1)}}{\Delta X} \right] e_{j+1,p,j+1}^{(1+1)} \\ & + \left[\frac{2^{\frac{\phi}{4}}}{(1-\frac{\phi}{4})} \ (\frac{-1}{(\Delta Y)^{2}} + \frac{V_{j+1,p,j+1}^{(1+1)}}{2(\Delta Y)}) \right] \ e_{j+1,p+1,j+1}^{(1+1)} \\ & = \frac{e_{j+1,p,1}}{\Delta^{\frac{1}{4}}} + \frac{U_{j+1,p,j+1}^{(1+1)}}{\Delta X} + \frac{e_{j+1,p,j+1}}{2(\Delta Y)} \right] c_{j+1,p-1,j+1}^{(1+1)} \\ & \left[\frac{1}{\Delta^{\frac{\phi}{4}}} + \frac{2^{\frac{\phi}{4}}_{R}}{8c(\Delta Y)^{2}} + \frac{2^{(1-\frac{\phi}{4})} \ V_{j+1,p,j+1}^{(1+1)}}{2(\Delta Y)} + \frac{U_{j+1,p,j+1}^{(1+1)}}{\Delta X} \right] c_{j+1,p,j+1}^{(1+1)} \\ & + \left[\frac{1}{2^{\frac{\phi}{4}}}} + \frac{2^{\frac{\phi}{4}}_{R}}{(\Delta^{\frac{1}{4}})^{2}} + \frac{V_{j+1,p,j+1}^{(1+1)}}{2(\Delta Y)} \right] c_{j+1,p+1,j+1}^{(1+1)} \\ & + \left[\frac{1}{2^{\frac{\phi}{4}}}} + \frac{e_{j+1,p,j}}{(\Delta^{\frac{1}{4}})^{2}} + \frac{V_{j+1,p,j+1}^{(1+1)}}{2(\Delta Y)} \right] c_{j+1,p+1,j+1}^{(1+1)} \\ & = \frac{c_{j+1,p,j}}{\Delta^{\frac{1}{4}}} + \frac{U_{j+1,p,j+1}^{(1+1)}}{2(\Delta^{\frac{1}{4}})} + \frac{U_{j+1,p+1,j+1}^{(1+1)}}{\Delta^{\frac{1}{4}}} \\ & + \left[\frac{1}{2^{\frac{\phi}{4}}} + \frac{e_{j+1,p,j}}{(\Delta^{\frac{1}{4}})^{2}} + \frac{V_{j+1,p,j+1}^{(1+1)}}{2(\Delta^{\frac{1}{4}})} \right] c_{j+1,p+1,j+1}^{(1+1)} \\ & = \frac{c_{j+1,p,j}}{\Delta^{\frac{1}{4}}} + \frac{U_{j+1,p,j+1}^{(1+1)}}{2(\Delta^{\frac{1}{4}})} \\ & + \frac{c_{j+1,p,j}}{\Delta^{\frac{1}{4}}} + \frac{U_{j$$

A constant mesh size of $\Delta X = 0.02$ was used with 50 steps in X-direction, till the upper edge of the plate (X = 1.0) was reached. The computational procedure took slightly over 3 minutes of CPU time for $U_{\infty} = 0.0$, Sc = 0.2 and N = 2.0 which reduced further with Sc = 2.0 and N = 0.0. For $U_{\infty} = 1.0$ and 10.0 CPU time required was around 1 minute. This time could have been shortened further if a variable mesh sizes were used in the X-direction, but in the interest of simplicity in computer programming constant mesh size was used in the X-direction.

Chapter 4

RESULTS AND DISCUSSION

4.1 Limiting Checks:

In order to assess the accuracy of the numerical procedure, several cases were solved for pure natural convection $(U_{\infty} = 0.0)$ and resulting results were compared with those of Callahan and Marmer [13]. Excellent agreement was obtained for steady state velocity, temperature and concentration profiles at X = 1.0 for Pr = 1.0, Sc = 0.7, 7.0 and N = 0.0, 1.0, 2.0. Transient Nusselt and Sherwood numbers for free convection $(U_{\infty} = 0.0)$ were also compared with the results of Callahan and Marner [13] for N = 0.0, 2.0 and Pr = 1.0 and for various values of Sc. Excellent agreement was found over the entire time interval. Based on these comparisons, it is felt that the present numerical procedure can predict both transient and steady-state results quite accurately. Below we present results for Pr = 0.7, Sc = 0.2 and 2.0, N = 0.0 and 2.0, and $U_{\infty} = 0.1$ and 10. The Pr and Sc values are representative of a large number of gases.

4.2 Velocity, Temperature and Concentration Profiles:

Fig. 4.1 shows the typical development of transient dimensionless X-component of velocity U at Pr = 0.7, Sc = 0.2 and N = 2.0 covering conditions ranging from almost pure forced convection ($U_{\infty} = 10.0$), combined flow with almost equally strong

natural and forced convection contributions ($U_{\infty}=1.0$), and pure natural convection ($U_{\infty}=0.0$). The profiles presented are those at the upper edge of the plate i.e. at X = 1.0. Numerical values are listed in tables 4.4 to 4.6. For free convection case i.e. at $U_{\infty}=0.0$, it is observed that the velocity increases continuously with time until at $T\simeq 2.40$ it reaches a maximum value and then it decreases slightly to the steady-state value at $T\simeq 2.30$ (with degree of accuracy of $E=10^{-3}$). The difference between the temporal maximum in the velocity profile and the steady-state value, however, is quite small and is imperceptible if shown on the figure. That is why it is not shown.

The phenomenon of temporal maximum in the velocity profile is somewhat surprising. It has been observed and discussed by several investigators for the problem of transient free convection on a vertical plate in the absence of mass transfer. Siegel [18] based on an approximate integral analysis, was apparently the first to predict such a behaviour. Later analysis by Gebhart [19], Hellums and Churchill [12] and Kleppe and Marner [20] all confirmed the findings of Siegel. Callahan and Marner [13] predicted such a phenomenon for the more complex problem involving simultaneous effect of heat and mass transfer in which N and Sc in addition to Pr. are the controlling parameters. The maximum velocity apparently occurs when the buoyancy forces in the fluid are largest, and it is clear that both the magnitude of the maximum velocity and the time at which it occurs are functions of these three parameters. However

the phenomenon of temporal maximum is not observed for combined free and forced convection ($U_{\infty}=1.0$) and almost pure forced convection ($U_{\infty}=10.0$). In these cases forced convection dominates the natural convection.

The time required to reach steady state velocity decreases as U_{∞} is increased. This is due to gradual masking of the natural convection by the forced convection. Figs. 4.1 and 4.2 show that for the same value of N, Pr and Sc the velocity boundary thickness decreases with increasing U_{∞} . This is also expected and is in line with the general behaviour of laminar boundary layers. Numerical values are listed in tables 4.1 to 4.6.

The effect of parameter N on the steady state velocity profile, again at X = 1.0, is shown in figs. 4.3 and 4.4 for Pr = 0.7 and Sc = 0.2 and 2.0 respectively for values of $U_{\infty} = 0.0$, 1.0 and 10.0. Clearly, the contribution of mass diffusion to the buoyancy force increases the maximum velocity significantly for low values of U_{∞} , both 0.0 and 1.0, though the increase is comparatively less for $U_{\infty} = 1.0$, for values of $U_{\infty} = 10.0$ and more (forced flow regime) this increase is almost insignificant.

A comparison of figs. 4.3 and 4.4 shows that the effect of the contribution of mass diffusion to the buoyancy force decreases for higher Schmidt numbers. This may be attributed to the fact that the rate of mass transfer in the fluid, which

in turn influences the buoyancy force, decreases as the Schmidt number increases. In the forced convection regime, however, increase in Schmidt number does not affect the maximum velocity as expected. Numerical values for the same are listed in tables 4.1 to 4.12.

Figs. 4.5 and 4.6 depict the development of transient dimensionless temperature and concentration profiles at X = 1.0 for Pr = 0.7, Sc = 0.2 and 2.0 respectively and N = 2.0 for pure natural convection case i.e. U = 0.0. Numerical values of temperature and concentration distributions are listed in tables 4.4 and 4.10. The temperature and concentration distributions in fig. 4.5, much like the velocity profile in fig. 4.1, increase to a maximum value at T ~ 1.20 and then decrease slightly to the steady state value at ₹ ~ 2.80. Similarly in fig. 4.6 for Sc = 2.0 the temperature and concentration distributions increase to a maximum value at $\tau \simeq 1.6$ and then decrease slightly to the steady state value at T 🗠 2.80. Though the maximum temperature and concentration profiles for Sc = 0.2 occur sooner than that for Sc = 2.0, the time at which the respective temperature and concentration profiles reach their steady state value is the same. Experimental data obtained by Goldstein R.J. and Eckert [21] and Klei for a vertical flat plate subjected to a step change in heat flux also verified this interesting overshoot phenomenon. As in the case of velocity profile the parameters Sc, N and Pr influence the

^{*} This does not imply that their transient behaviour is identical.

extent of overshoot in the temperature and concentration profiles and the instant at which this maximum occurs. However this overshoot phenomenon is not observed when forced convection in addition to natural convection comes into play.

A comparison of the transient concentration profiles in figs. 4.5 and 4.6 shows that the concentration boundary layer for pure natural convection $(U_{\infty}=0.0)$ is considerably thinner for Sc = 2.0 than that for Sc = 0.2. But for $U_{\infty}=10.0$ i.e. when forced convection is dominant, this difference in the concentration boundary layer thickness at Sc = 0.2 and 2.0 is much less as seen from fig. 4.7 and table 4.6. Similar conclusions can be drawn for N = 0.0 by comparing figs. 4.9 and 4.10.

Figs. 4.8 and 4.9 show the steady state temperature and concentration profiles at X = 1.0 for Pr = 0.7, Sc = 2.0 and N = 2.0 and 0.0 respectively for all three values of U_{∞} . From figs. 4.5, 4.6, 4.3 and 4.9 comparing transient and steady state temperature and concentration profiles for various values of N_{\star} Sc and V_{∞} it is observed that the concentration boundary layer is thicker than thermal boundary layer for Sc = 0.2 while opposite is true for Sc = 2.0. This difference in thickness decreases as V_{∞} increases. Fig. 4.10 shows the steady state concentration profiles at $V_{\infty} = 1.0$, $V_{\infty} = 0.7$ and $V_{\infty} = 0.2$ with $V_{\infty} = 0.7$ and $V_{\infty} = 0.2$ with $V_{\infty} = 0.7$ and $V_{\infty} = 0.7$ and $V_{\infty} = 0.2$ with $V_{\infty} = 0.7$ and $V_{\infty} = 0.7$ and $V_{\infty} = 0.0$, $V_{\infty} = 0.0$, is increased. At $V_{\infty} = 10.0$,

^{*}The correct parameter is the Lewis number, Le = Sc/Pr, but Pr is held constant here.

both values of N yield almost the same concentration profile.

This is expected since buoyancy forces are insignificant at such a high forced flow condition. Numerical values are listed in tables 4.1 to 4.12.

4.3. Nusselt and Sherwood Numbers:

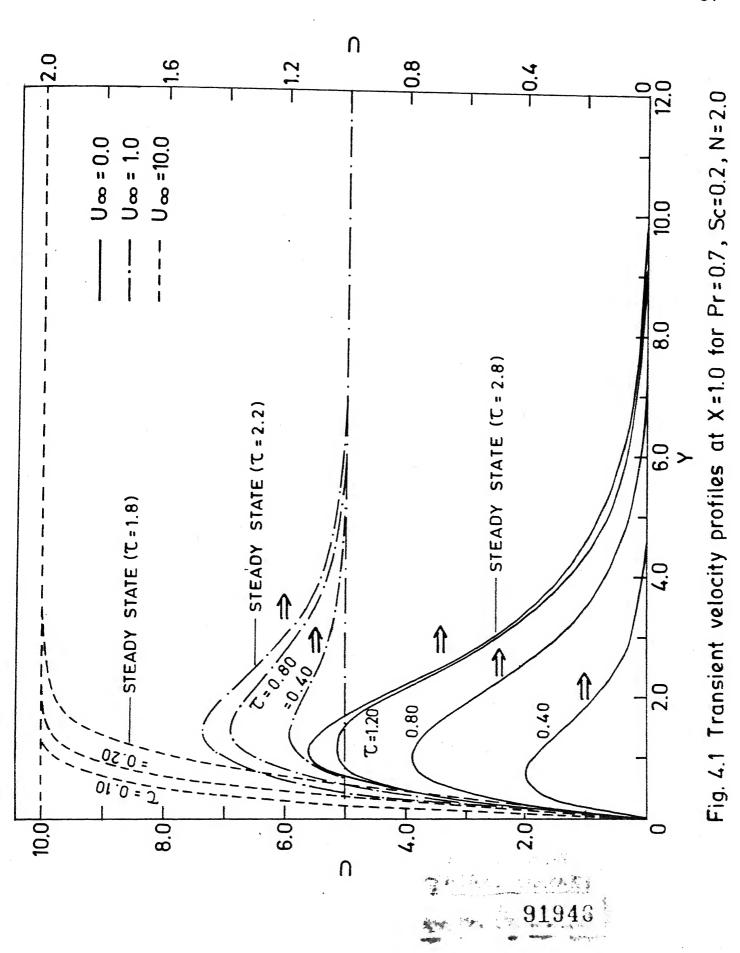
Transient mean Nusselt and Sherwood numbers are shown in figs. 4.11, 4.12 and 4.13 for Pr = 0.7, Sc = 0.2 and 2.0, N = 0.0 and 2.0, and U_{∞} = 0.0, 1.0 and 10.0. Numerical values are listed in tables 4.13 and 4.14. Initially for any set of parameters values of mean Nusselt number and Sherwood number are high but they drop drastically with time and approach steady state values. During this initial transient Um has negligible influence on both $\mathrm{Nu_{m}}$ and $\mathrm{Sh_{m}}$ due to the fact that heat is transferred by conduction only and mass is transferred by diffusion only during this regime. As the buoyancy forces due to mass transfer and thermal convection increase, the velocity increases sufficiently for N and \mathbf{U}_{∞} to influence the solution. Because of the overshoot phenomenon observed in the temperature and concentration profiles (i.e. these profiles reaching a maximum before steady-state conditions are reached), a transient minimum is observed in both the Nusselt and Sherwood numbers, for pure natural convection. The difference between the temporal minimum and the steady state value, however, is quite small, and in most cases is nearly imperceptible on the figures. (See the inset in fig. 4.12). However for $U_{\infty}=1.0$ and 10.0 where forced convection is predominant such a temporal minimum is not observed since the overshoot phenomenon is not present.

Both the mean Nusselt and the mean Sherwood numbers show a slight dependence on Sc and U_{∞} as far as the time required to reach steady state conditions is concerned, i.e., as Sc increases and as U_{∞} decreases the time decreases. However, the time required to reach steady-state conditions is virtually insensitive to the parameter N.

As the Schmidt number increases from 0.2 to 2.0 with Prandtl no. fixed at 0.7 and N fixed at 2.0, Sherwood number increases substantially but the Nusselt number is only moderately affected.

From figs. 4.14 and 4.15 it is noted that an increase in N results in higher Nusselt and Sherwood numbers. That is, an increase in the buoyancy force due to mass transfer results in an increase in the rates of heat and mass transfer. The effect of parameter N on the Nusselt number becomes less pronounced with increasing Schmidt number but its effect on Sherwood number is opposite and to a small extent. This behaviour can be explained as follows. It was observed that an increase in Schmidt number decreases the concentration boundary layer thickness. However, the thermal boundary layer thickness is relatively less sensitive to an increase in the Schmidt number for fixed values of Pr and $\rm U_{\infty}$ eventhough the conservation equations are coupled. Thus with

an increase in Schmidt number the concentration boundary layer thickness becomes thinner than the thermal boundary layer thickness. Hence the influence of parameter N on Num reduces with increasing, Schmidt number, as the buoyancy effect due to mass transfer are diminished in the thermal boundary layer. On the contrary thermal buoyancy effects become less important as compared to buoyancy effects due to mass transfer in the thinner concentration boundary layer. Hence the influence of N on Shm increases with increased Schmidt number.



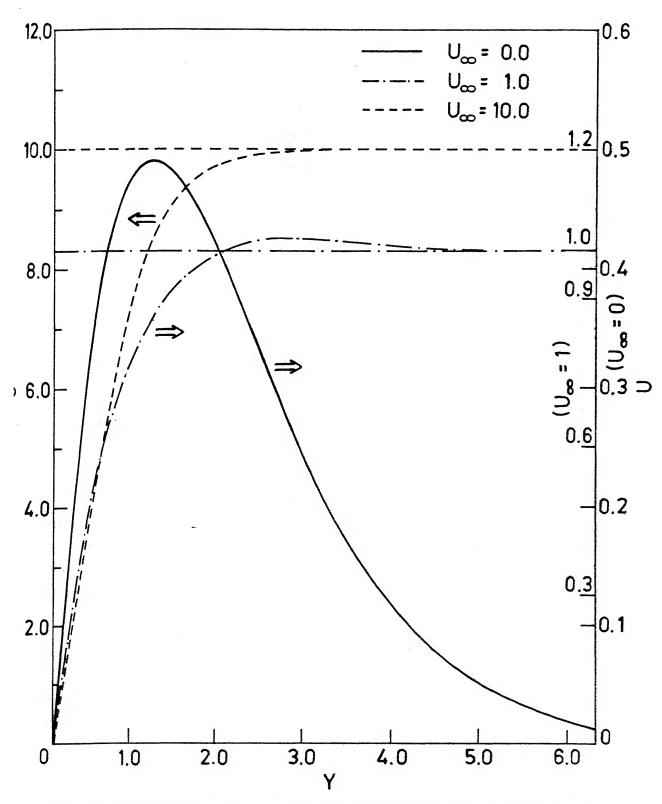


Fig. 4.2 Steady state velocity profiles at X = 1.0 for Pr = 0.7, Sc = 0.2, N = 0.0

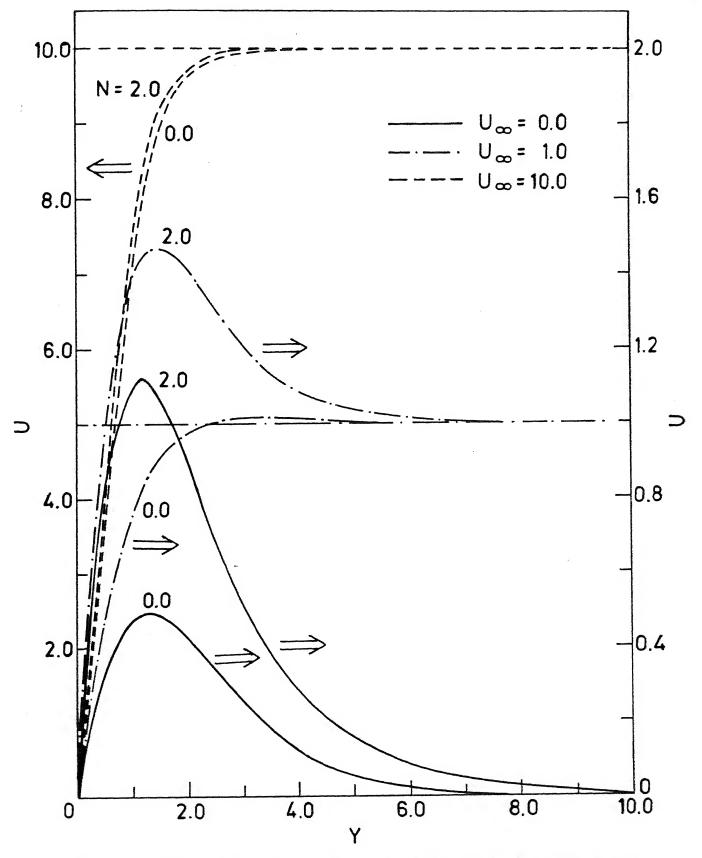


Fig. 4.3 Steady state velocity profiles at X = 1.0 as a function of N for Pr = 0.7, Sc = 0.2

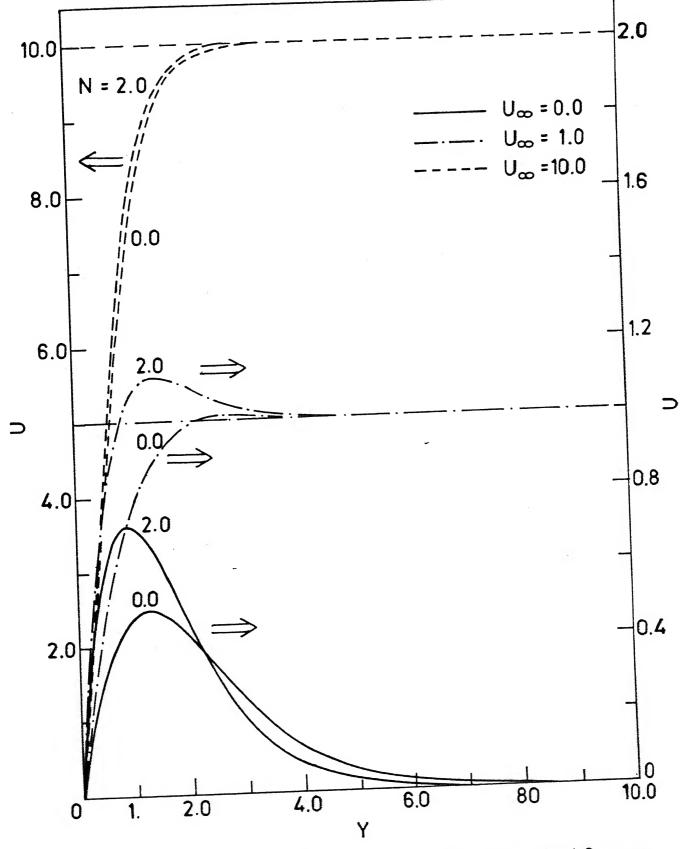


Fig. 4.4 Steady state velocity profiles at X = 1.0 as a function of N for Pr = 0.7, Sc = 2.0

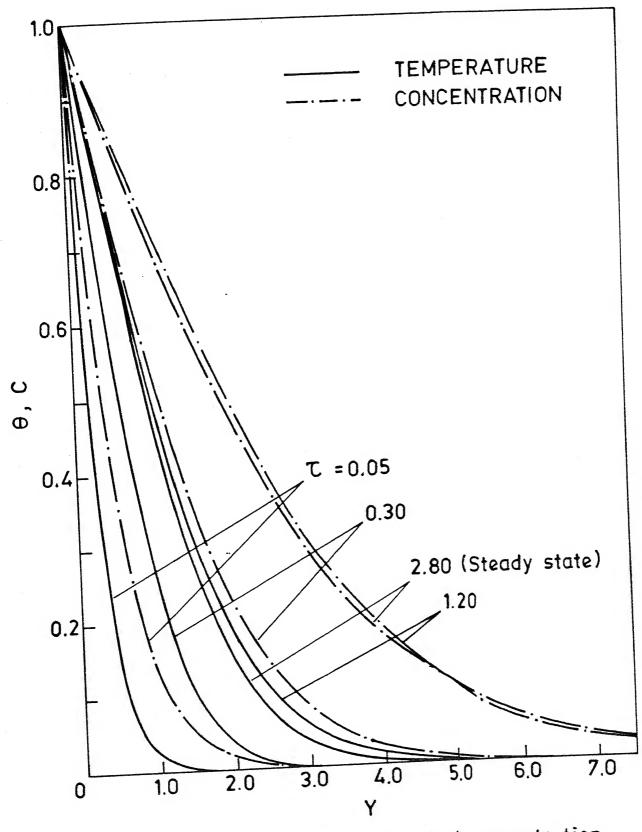


Fig. 4.5 Transient temperature and concentration profiles at X = 1.0 for Pr = 0.7, Sc = 0.2, N = 2.0 and U_{∞} = 0.0

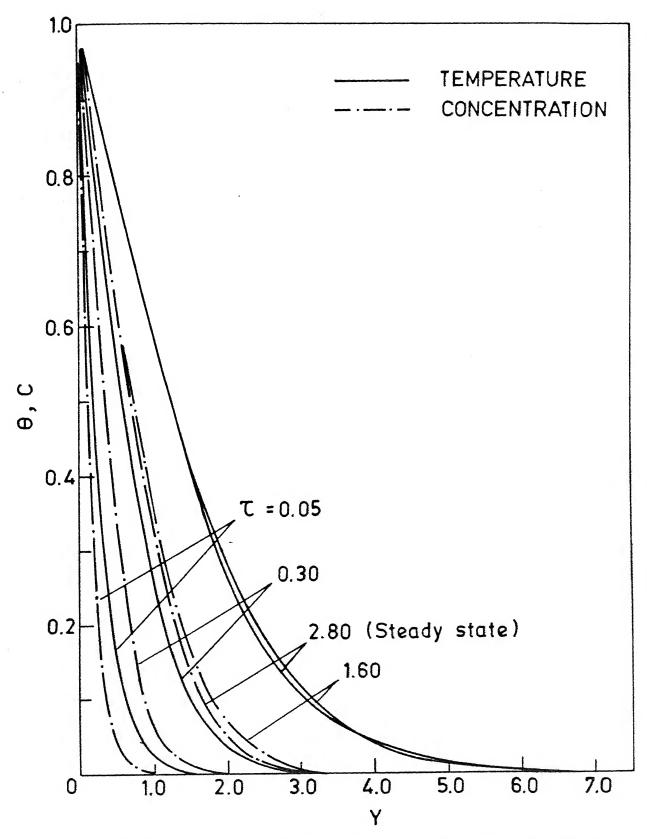


Fig. 4.6 Transient temperature and concentration profiles at X = 1.0 for Pr = 0.7, Sc = 2.0 , N = 2.0 and U_{∞} = 0.0

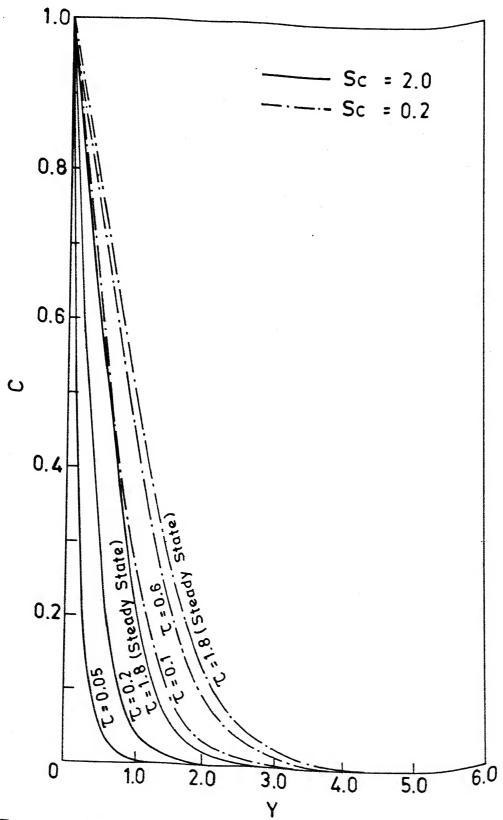


Fig. 4.7 Transient concentration profiles at X=1.0 for Pr=0.7, N=2.0, Sc=0.2 and 2.0 and $U_{\infty}=10.0$

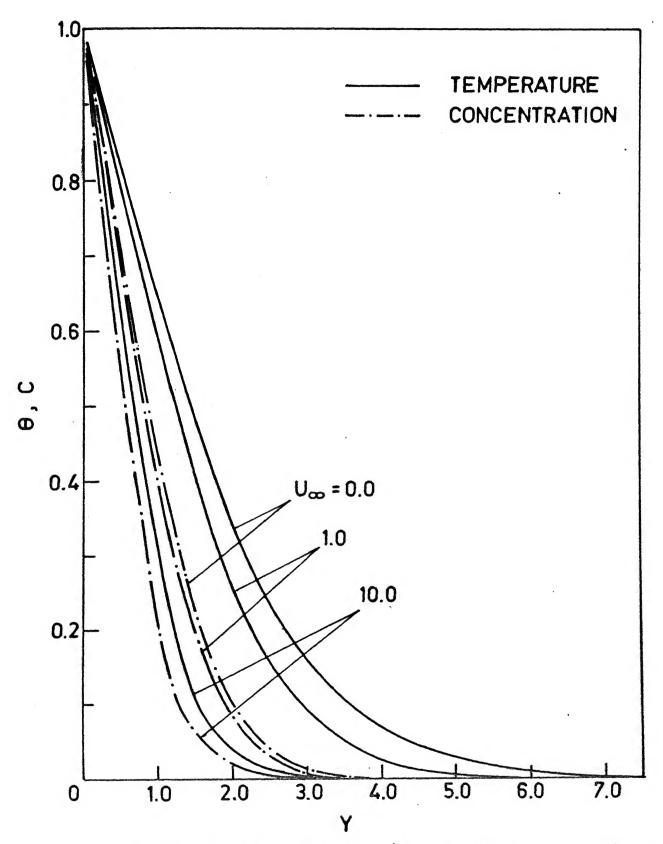


Fig. 4.9 Steady state temperature and concentration profiles at X = 1.0 for Pr = 0.7, Sc = 2.0, N = 0.0

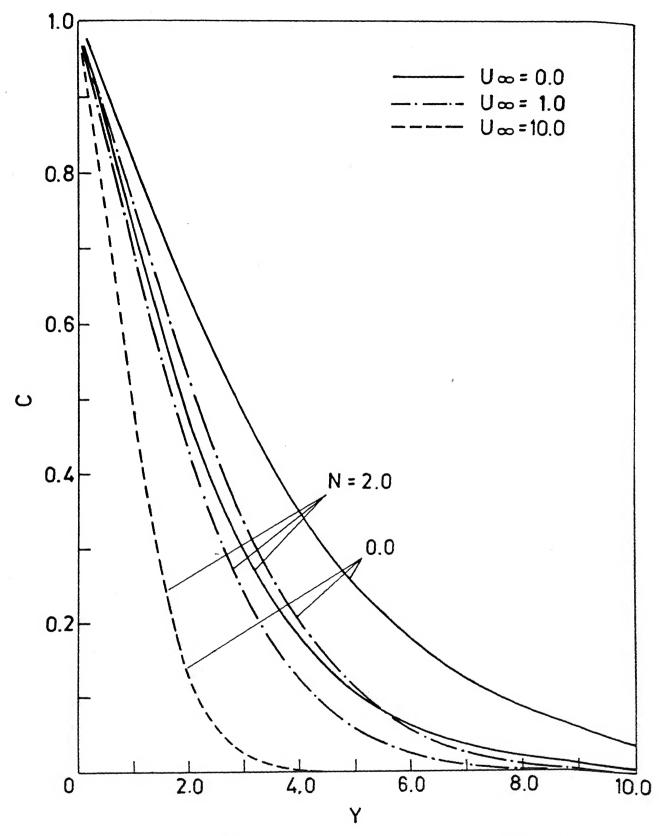


Fig. 4.10 Steady state concentration profiles at X = 1.0 as a function of N for Pr = 0.7, Sc = 0.2

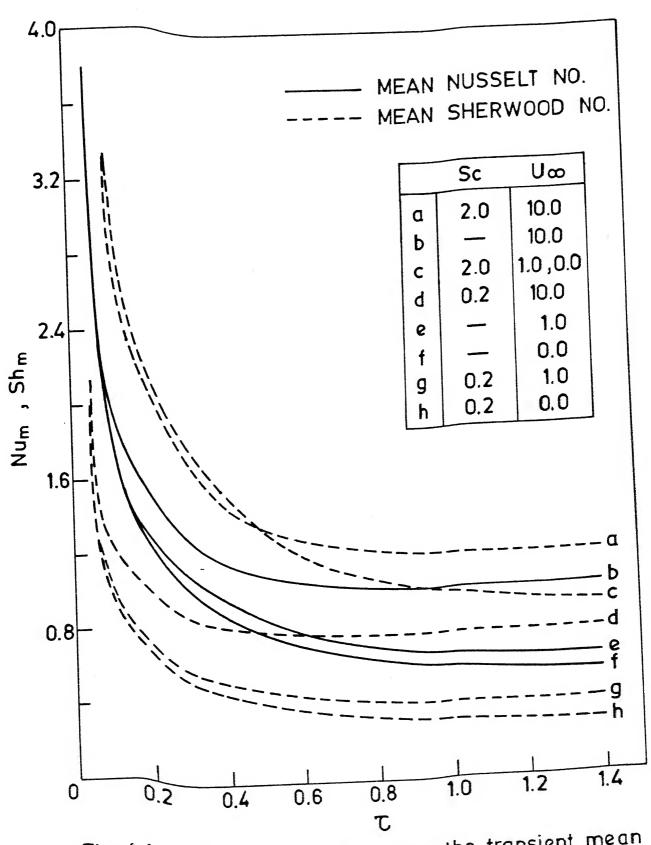


Fig. 4.11 Effect of Sc and U_{∞} on the transient mean Nusselt and Sherwood nos. for Pr=0.7 and N=0.0

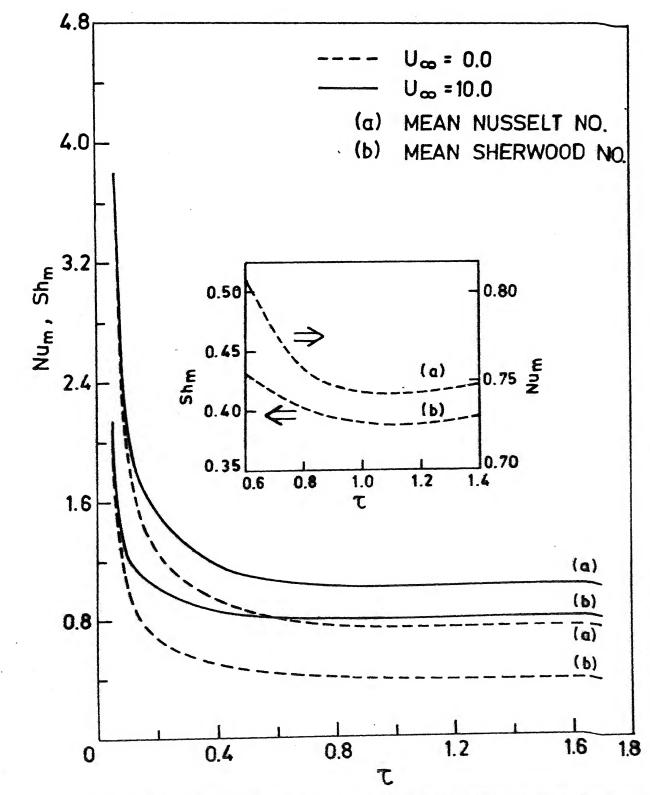


Fig. 4.12 Effect of U∞ on the transient mean Nusselt and Sherwood nos. for Pr=0.7, Sc=0.2 and N=2.0

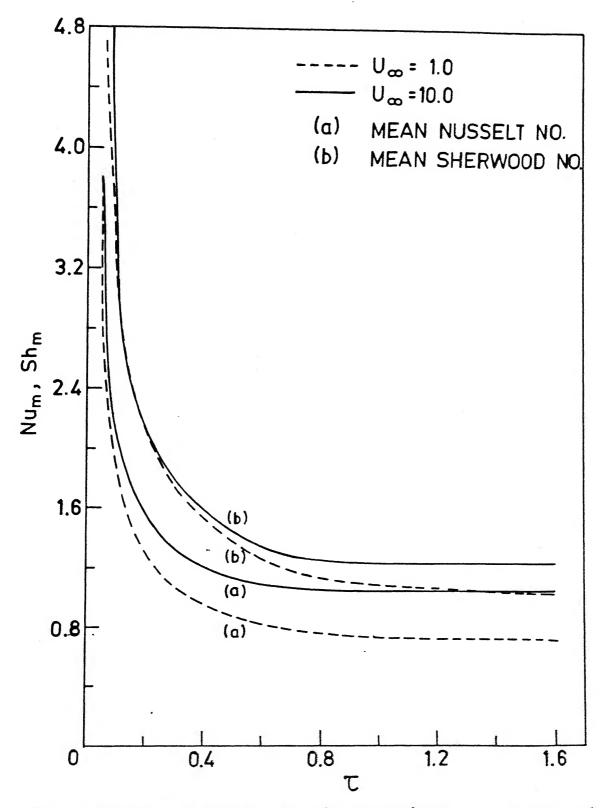


Fig. 4.13 Effect of U_{∞} on the transient mean Nusselt and Sherwood nos. for Pr=0.7, Sc=2.0, N=2.0

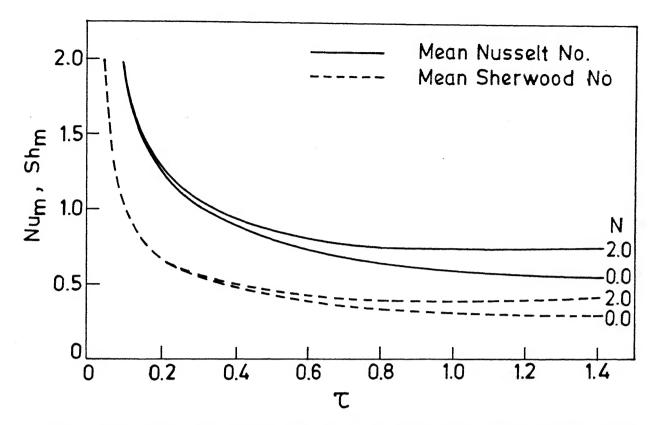


Fig. 4.14 The effect of N on the transient mean Nusselt and Sherwood numbers for Pr = 0.7, Sc = 0.2, $U_{\infty} = 0.0$

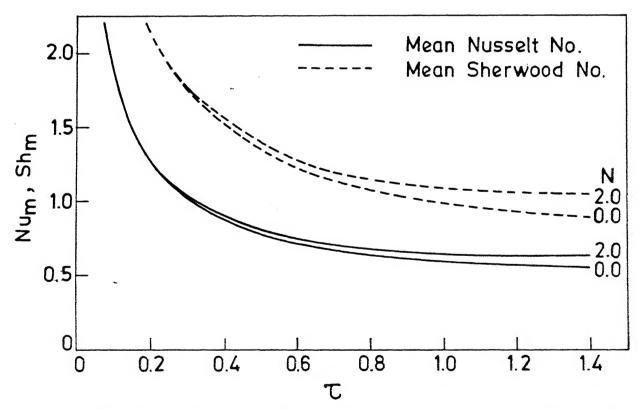


Fig. 4.15 The effect of N on the transient mean Nusselt and Sherwood numbers for Pr = 0.7, Sc = 2.0, $U_{\infty} = 0.0$

Fine required to reach steady state = 3.2

Table 4.1 : Steady state Velocity , Lemberature § Concentration distributions at k=1.0 for Pr=0.7,Sc=0.2,%=0.0 and % =0.0

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Table 4.2: Steady state velocity, Temperature () Concentration distributions at for Pr=0.7, Sc=u.2, N=0.0 and 0 =1.

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Table 4.4 b : Tau = 0.3 : Velocity, Temperature and Concentration distribution at X = 1.0 for Pr = 0.7.Sc = 0.2.7 = 2 and $T_{\infty} = 0.0$.

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Table 4.4 c: Tau = 0.4 : Velocity, Temperature and Concentration distribution at X = 1.0 for Dr = 0.7, Sc = 0.2, Y = 2 and $U_{\infty} = 0.0$.

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MENN MISSELT HO= .7573 MEAN SHERWOOD ID= .4039
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Table 1.1 d : Tau = 0.8 : Velocity, Temperature and Concentration distribution at X = 1.0 for Pr = 0.7, Sc = 0.2, Y = 2 and $Y_{\infty} = 0.0$.

k	Y	u	Λ	THET	Cha
11123456789012345678901234567890123456789012345678 1111111111222222222222333333333444444444		-220155544769862296217644428921488289778658585611285778612295116099886295778612221160998862957786122231550577893030998862862741197664332211000000000000000000000000000000000	-1.06756624 -1.067356624 -1.06735624 -1.06735624 -1.06735624 -1.067357 -1.067357 -1.76904 -1.776	\$\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\	3.3696 3.3736 3.3796

Table 4.4 e: Tau = 1.2: Welocity, Tamparature and Concentration distribution at x = 1.0 for pr = 0.7, Sc = 0.2, M = 2 and N_∞ = 0.0.

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K
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0.3250
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0.73860
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                                                 1.848 SHERWOOD HO= .3973
WEAM AUSSELF HO= .7589
```

Table 4.4 f: Steady state Velocity , lemperature & Concentration distributions at x=1.0 for Pr=0.7, Sc=0.2, Y=2.0 and Ω_{∞} =0.0

Time required to reach steady state = 2.8

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MEAN NUSSELT NU=1.0057 MEAN SHERWOOD NO= .5775
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Table 1.5 a: Tau = 0.4: Velocity, Temperature and Concentration distribution at X = 1.0 for pr = 0.7, Sc = 0.2, M = 2 and M = 1.0.

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Table 4.5 b : Tay = 0.8 : Velocity, lemberature and Concentration distribution at X = 1.0 for Pr = 0.7, Sc = 0.2, N = 2 and $U_{\infty} = 1.0$.

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Table 4.5 c: Steady state velocity , Temperature & Concentration distributions at X=1.0 for Pr=0.7, Sc=0.2, N=2.0 and 0 =1.0

to reach steady state

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and Concentration distribution at X = 1.0 for pr = 0.7,Sc = 0.2,N = 2 and U.
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MEAN NUSSELT NO= .6298 NFAN SHERWOOD NO= .8829 Time required to reach steady state = 2.0

Table 4.8 :Steady state Velocity, Temperature, Concentration distributions at X=1.4 for Pr=0.7, Sc=2.6, N=0.0 and U =1.0

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MEAN NUSSELF NO=1.0192 MEAN SHERWOOD NO=1.1980 Time required to reach steady state = 2.0

Table 4.9 :Steady state Velocity, Temperature & Concentration distributions at X=1.0 for Pr=0.7, Sc=2.0, N=0.0 and Um=10.0

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Table 4.10 c : Tau = 1.60 : Velocity, Temperature and Concentration distribution at X = 1.0 for Pr = 0.7, Sc = 2.0, N = 2 and U = 0.0 .

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                                 SHERWOOD
                                             47=1.961d
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Pime required to reach steady state

Steady state velocity, Temperature and Concentration distributions a X=1.0 for Pr=0.7, Sc=2.0, N=2.0 and U_ =0.0 Table 4.10 d: t œ

					- 2	
K	Y	IJ	v	PHETA	CONC	
112345678901234567890123456789012 1111111111222222222233333333333444 A	369258147000000000000000000000000000000000000		-0.5833 103 AFAV	493827161658752254047036036727897249504100 0 0 1 493827161658752254047036036727897249504100 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	2865321966182957196270775797038478426000000000000000000000000000000000000	69
rine r	equire	d to re	ach stea			

Table 4.11 :Steady state Velocity, Temperature and Concentration distributions at x=1.0 for Pr=0.7,Sc=2.0,M=2.0 and =1.0

Y U V YEART	######################################	Table 4.12 b: Tau = 0.2: Velocity Temperature X = 1.0 for Pr = 0.7; Sc = 2.0; N = 2 and U = 10.0
A LHETA CONC	12.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.	Table 4.12 a : Tau = 0.05 : Velocity Femerature x = 1.0 for pr = 0.7,5c = 2.0,4 = 2 and 100 = 10.0

and concentration distribution at and 0.00 = 10.0.

K	Y U	<u>V</u>	THETA	conc
12345678901234567890123456789 112345678901234567890123456789 11111111112233333333333333333333333333		00000000111111111111111111111111111111	370471471483730889426017229186805141000 0 9879654333222110011220276116846805141000 0 99999999998888774185258815050831031000000 0 00000 0000 0000 0000 0000	997654324334451113619668617620537845189621060 9976543243344577766055543728550232957578621060 9999999999999998877776666554332211100575190600000000000000000000000000000000000
Time requir	ed to read	n steady	state	= 1.80

Paole 4.12 c :Steady state Velocity, Tencerature and Concentration distributions at X=1.0 for Pr=0.7, Sc=2.0, 4=2.0 and U =10.0

	TAU	MEAN NUSSELT	MFAN SHEKWUND NG.	TAU EUSSFUT SUBLICAGO MU. 10.
10 10 10 10 10 10 10 10 10 10 10 10 10 1	50505050000000000000000000000000000000	077749593 88403497369195 7814511551981244989901334 6942110997665555555555555555555555555555555555	11600356229250690038394 907655189459225118377766 9076555433332222222222222222222222222222222	11.75 0 4 7 5 0 4 7 7 5 0 4 7 7 5 0 4 7 7 5 0 4 7 7 5 0 4 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7
U _∞ =1.∪	05050505000000000000000000000000000000	31.69249806555475798 1.9498571785542519 1.0978176543519 1.0900.666666666666666666666666666666666	2.0849124601000963 0.087124601000963 0.0875637963100096 0.080000000000000000000000000000000000	0.000 0.000
U _∞ =10	01050000000000000000000000000000000000	79.239.41 79.239.41 10.6544.61 10.0324.65 10	21110000000000000000000000000000000000	0 = 1

for N = 0.0

for : = 2.0

Table 4.13: Transfert mean Nusselt and Enerwood Dymbers for Pr = 0.7,Sc = 0.2

	TAU	MEAN MUSSEUT	MEAN SHERADUD	TAH NUSSELL SHERWOOD
J = () . ()	5 05 05 05 00 00 00 00 00 00 00 00 00 00	31111110000000000000000000000000000000	115391115358197614204 990204353905209889995 02519765200999889995 632211111110000000000	100 1 1 1 1 1 3 3 3 1 3 4 7 7 9 8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
J ₂ =1.0	2 01-223m+6872+680 01-223m+6872+680 1-2123m+6872+680	0.54-5 6924-9896554-75798-5-04-216715-185-04-2519 311-1111-0-0-0-0-0-0-0-0-0-0-0-0-0-0-0-0	0 - 8 - 4 - 4 - 6 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1	1.0
U ₂₀ =10.	00000000000000000000000000000000000000	3.7913 2.06314 1.0701 1.0701 1.03140 1.02093 1.0193 1.0193 1.0193	6.17379 113779 124279 114379 1	Tello 37932 0 13050 1 1
	for .	4 = 9.0		for a = 2.0

Table 4.14: Transient mean ausselt and Sherwood numbers for Pr = 0.7, Sc = 2.0

Chapter 5

CONCLUSIONS

A boundary-layer analysis for transient, laminar, combined forced and natural convection along an isothermal vertical flat plate subjected to a step change in temperature and concentration has been solved by a highly implicit finite-difference method. In order to obtain a solution to this problem, the coupled governing conservation equations must be solved simultaneously. The computer code in the Appendix is based on a non-uniform mesh in the direction normal to the plate (Y-direction). The parameters of the problem are:

(i) the buoyancy ratio parameter, N, (ii) the Prandtl number, Pr, (iii) the Schmidt number, Sc, and (iv) the Forced-free convection parameter, $U_{\infty} = \text{Re}/\text{Gr}^{1/2}$. Results found for various values of the above parameters show the following:

- a) During the initial transient period the heat transfer is by conduction only, and the mass transfer is by diffusion only, even for strong forced flow.
- b) After the initial conduction-diffusion regime, combined buoyancy forces along with free stream velocity (for combined forced and free flow) generate the motion.
- c) The transient velocity, temperature and concentration profiles for free convection show a temporal maximum

over their respective steady state values.

However, this phenomenon of temporal maximum is not observed for combined free and forced convection.

- d) The time required to reach the steady state decreases with increase in forced-free convection parameter \mathbf{U}_{∞}
- e) Both Nusselt and Sherwood numbers pass through a temporal minimum before reaching their steady state values. However, it is observed only for free convective flow and not for combined free and forced convection.
- f) For mass diffusion aiding the flow both mean Nusselt and Sherwood numbers are higher than those for N = 0.0 (Pure thermal convection).
- g) Mean Nusselt and Sherwood numbers are higher for higher values of U_{∞}

APPENDIX

COMPUTER PROGRAM LISTING

```
***********************************
C for transient, laminar, free-forced convection with heat and
C mass transfer from a isothermal plate.
Variables:-
C.
C
    A(I)
           lower diagonal elements in a TRIDIAGONAL matrix
C
           main diagnal elements in TRIDIAGONAL matrix
   B(I)
C
           urrer diasonal elements in TRIDIAGONAL matrix
   C(I)
C
   AU, AV, ATHET, ACON
                          : suess values in the iterative method
C
                          : previous x-location values
   PU, PV, PTHET, PCON
                          : values at the previous time
C
   UI, THETI, CONI
                   instantaneous local Nusselt and Sherwood
C
   NUX,SHX
С
                   numbers respectively
                   instantaneous mean Nusselt and Sherwood
C
   NUM, SHM
                   numbers respectively
C
C
   PR
           Pradtl number
C
   SC
           Schmidt number
C
   RN
           buoyancy ratio parameter
C
           free-forced convection Parameter
   UINF
C
           relaxation factor
   DELY1, DELY2, DELY3, DELY4 : step sizes n Y direction
C
C
                          : ratio of smaller step size to
   ALPHI
C
                            larger step size in Y direction
C
   DELX
                          : step size in X direction
C
                          : time-sters
   DETAU1, DETAU2
           number of steps in X direction
C
           number of steps in Y direction
C
C
                  Points at chanse of mesh size
   IP, IQ, IR
C
   REAL NUX, NUM
   DIMENSION A(39),B(39),C(39),D(39),U(39),THET(39),AU(39)
   DIMENSION ATHET(39), PU(39,51), PTHET(39,51), V(39), AV(39)
   DIMENSION CON(39),ACON(39),PCON(39,51),Y(39)
   DIMENSION UI(39,51), THETI(39,51), CONI(39,51)
   DIMENSION NUX(51),SHX(51)
   OPEN (UNIT=21,DEVICE='DSK',FILE='FFFF,CDR')
   OPEN (UNIT=25,DEVICE='DSK',FILE='FFCC16,DAT')
   OPEN (UNIT=23,DEVICE='DSK',FILE='N816,DAT')
C
C Read in and print out the parameters
```

```
READ(21,*)N, IP, IQ, IR, PR, THEO, SMALL, SC, RN, CONO, SSMALL READ(21,*)DELVI
\Gamma
          READ(21,*)DELY1,DELY2,DELY3,DELY4,DTAU1,DTAU2
          WRITE(25,111)N,M,DELX,DELY1,DELY2,DELY3,DELY4,DTAU1,DTAU2
111 FORMAT(2X, 'NO OF STEPS IN Y DIRECTION = ',13,'
           1,2X,'NO OF STEPS IN X DIRECTION = ',13,/
           3,2X, STEP SIZES IN Y DIRECTION = ',F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4.2,1X,F4
           4,F4.2,/,2X,'TIME STEP = ',F4.2,2X,F4.2,'8',F4.2,2X
5,'CONUFRENCE COTTON:
           5, CONVERGENCE CRITERIA IS EPS LESS THAN (,F7.5)
          FORMAT(2X, 'For', 3X, 'Prandtl NO. =',F4.2,2X', c4.2
           1, Schmidt NO. = 1, F4.2, \frac{2}{7}, F4.2
2, \frac{2}{7}, F4.2
2, \frac{2}{7}
94
           2,2X,737,2X,78e/(Gr)**0.5 = 7,F5.2
           WRITE(25,95)
888 FORMAT(4X, 'TAU', 5X, 'MEAN NUSSELT NO.', 4X, 'MEAN SHERWOOD NO.')
WRITE(23,887)
        Values of all the constants required to be c^{lpha l}culated repeatedly calculated here
887 FORMAT(2X,50(1H-))
ж
ж
        calculated here
                DY2=0.5/DELY2;DY1=0.5/DELY1;DTAU=DTAU1;DT=1.0/DTAU
DY3=0.5/DELY2;DY1=0.5/DELY1;DTAU=DTAU1;DT=1.0/DTAU
_{*}
ж
                DELX=1./DELX;D1=DELY1*DELX;D2=DELY2*DELX;D3=DELY3*DELXD4=DELY4*TF1Y
                DYM2=4.0*DYZ*DYZ;DYM1=4.0*DY1*DY1;DYE2=DYMZ/PR;DYE4=DYM4/PR
DYM3=4.0*DYZ*DYZ;DYM4-4.0*DY1*DYE2=DYMZ/PR;DYE4=DYM4/PR
                DYM3=4.0*DY3*DY3;DYM4=4.0*DY4*DY4;DYE3=DYM3/PR;DYE4=DYM4/PR
DYC2=DYM2/SC:DYC1-DYM1/CC
                DYC2=DYM2/SC;DYC1=DYM1/SC
                APHI1=2.*APHI*APHI/(1.0+APHI);APHI2=(APHI-1.)/(APHI+1.)
APHI3=2.*(1.-APHI)
                BPHI3=2.**/ - PERIT BPHI/(1.0+BPHI);BPHI2=(BPHI-1.)/(BPHI+1.)
                CPHI1=2.*CPHI*CPHI/(1.0+CPHI);CPHI2=(CPHI-1.)/(CPHI+1.)
CPHI3=2.*(1.-CPUT)
                 CPHI3=2.*(1.-CPHI)
                 TAU=0.0
                 IP2=IP+1;IQ2=IQ+1;IR2=IR+1
                 Y(1) = DELY1
```

```
DO 180 K=2,IP
       Y(K)=Y(K-1)+DELY1
180
       CONTINUE
       DO 190 K=IP2,IQ
       Y(K)=Y(K-1)+DELY2
190
      CONTINUE
      DO 191 K=IQ2,IR
      Y(K)=Y(K-1)+DELY3
191
      CONTINUE
      DO 192 K=IR2,N
      Y(K)=Y(K-1)+DELY4
192
      CONTINUE
   initial and boundary conditions are generated and initial guesses
ж
   for U's,V's,Theta's and C's at 1,an iteration counter ,in the
   iterative procedure (cf. section 3.3) are siven
ж
    DO 10 K=1 v N
    AU(K)=UINF;AU(K)=0.0;ATHET(K)=EXP(-1.9*Y(K))
    ACON(K) = EXP(-1.9*Y(K))
    PH(K = 1) = HINF # PTHET (K = 1) = 0.0 # PCON(K = 1) = 0.0
    DO 10 L=1, M
    UI(K,L)=UINF;THETI(K,L)=0.0;CONI(K,L)=0.0
10
    CONTINUE
C
C
   marching in time
C
125 TAU=TAU+DTAU
    WRITE(5,*)TAU
C
C
   for higher times relaxation factor - ALP is increased
C
    IF(TAU.GE.1.)ALP=1.0
C
   marching in X - direction (downstream)
    J=0 $X=0.0
    DO 140 L=1 M
    X=X+1.0/DELX
    J=J+1
    ITR=0
400 ITR=ITR+1
* elements of TRIDIAGONAL coefficient matrix and right hand side
* vector for momentum equation (cf. eqn. 3.17)are calculated
*
```

```
DO 20 K=1, IP
    PROD=AV(K)*DY1
    A(K)=-DYM1-PROD
     B(K)=AU(K)*DELX+2.0*DYM1+DT
    C(K)=-DYM1+PROD
20
    CONTINUE
    A(IP)=A(IP)+APHI2*C(IP)
    B(IP)=B(IP)+APHI3*C(IP)
    C(IP)=C(IP)*APHI1
来
ж
   step size is changed from smaller step size DELY1 to larger step
来
   size DELY2
    DO 30 K=IP2,IQ
    PROD=AV(K)*DY2
    A(K) = -DYM2 - PROD
    B(K)=AU(K)*DELX+2.0*DYM2+DT
    C(K) = -DYM2 + PROD
30
    CONTINUE
    A(IQ)=A(IQ)+BPHI2*C(IQ)
    B(IQ)=B(IQ)+BPHI3*C(IQ)
    C(IQ)=C(IQ)*BPHI1
*
   step size is chansed from smaller step size DELY2 to larser step
*
   size DELY3
^*
ж
    DO 21 K=IQ2,IR
    PROD=AV(K)*DY3
    A(K)=-DYM3-PROD
    B(K)=AU(K)*DELX+2.0*DYM3+DT
    C(K)=-DYM3+PROD
   CONTINUE
21
    A(IR)=A(IR)+CPHI2*C(IR)
    B(IR)=B(IR)+CPHI3*C(IR)
    C(IR)=C(IR)*CPHI1
ж
   ster size is chansed from smaller ster size DELY3 to larser ster
求
Ж.
   size DELY4
來
    DO 31 K=IR2,N
    PROD=AV(K)*DY4
    A(K) = -DYM4 - PROD
    B(K)=AU(K)*DELX+2.0*DYM4+DT
    C(K)=-DYM4+PROD
```

```
size DELY2
×.
ж
    DO 90 K=IP2,IQ
    PROD=V(K)*DY2
    A(K)=-DYE2-PROD
    B(K)=U(K)*DELX+2.0*DYE2+DT
    C(K)=-DYE2+PROD
90
    CONTINUE
    A(IQ) = A(IQ) + BPHI2 *C(IQ)
    B(IQ)=B(IQ)+BPHI3*C(IQ)
    C(IQ)=C(IQ)*BPHI1
¥.
來
   step size is chansed from smaller step size DELY2 to larger step
Ж
   size DELY3
ж
    DO 81 K=IQ2,IR
    PROD=V(K)*DY3
    A(K)=-DYE3-PROD
    B(K)=U(K)*DELX+2.0*DYE3+DT
    C(K)=-DYE3+PROD
    CONTINUE
81
    A(IR) = A(IR) + CPHI2 * C(IR)
    B(IR)=B(IR)+CPHI3*C(IR)
    C(IR)=C(IR)*CPHI1
*
*
   step size is chansed from smaller step size DELY3 to larger step
   size DELY4
×
^*
    DO 91 K=IR2,N
    PROD=U(K)*DY4
    A(K)=-DYE4-PROD
    B(K)=U(K)*DELX+2.0*DYE4+DT
    C(K)=-DYE4+PROD
91
    CONTINUE
    DO 100 K=1,N
    D(K)=U(K)*FTHET(K,L)*DELX+THETI(K,L)*DT
100 CONTINUE
    D(1) = D(1) - A(1) \times THEO
    CALL TRIDI (A,B,C,THET,D,N)
*
菜
  calculation of Theta's (temperatures) at various Y-locations
  is complete
* elements of TRIDIAGONAL coefficient matrix and right hand side
* vector for species equation (cf. eqn. 3.19) are calculated
```

```
*
     DO 150 K=1, IP
     PROD=V(K)*DY1
     A(K) = -DYC1 - PROD
     B(K)=U(K)*DELX+2.0*DYC1+DT
     C(K) = -DYC1 + PROD
150 CONTINUE
     A(IP)=A(IP)+APHI2*C(IP)
     B(IP)=B(IP)+APHI3*C(IP)
     C(IP)=C(IP)*APHI1
*
¥.
   ster size is changed from smaller ster size DELY1 to larger ster
Ж
   size DELY2
*
    DO 160 K=IP2,N
    PROD=V(K)*DY2
    A(K) = -DYC2 - PROD
    B(K)=U(K)*DELX+2.0*DYC2+DT
    C(K) = -DYC2+PROD
160 CONTINUE
    A(IQ)=A(IQ)+BPHI2*C(IQ)
    B(IQ)=B(IQ)+BPHI3*C(IQ)
    C(IQ)=C(IQ)*BPHI1
*
*
   step size is changed from smaller step size DELY2 to larger step
   size DELY3
*
ж
    DO 151 K=IQ2,IR
    FROD=V(K)*DY3
    A(K) = -DYC3 - PROD
    B(K)=U(K)*DELX+2.0*DYC3+DT
    C(K) = -DYC3+PROD
151 CONTINUE
    A(IR)=A(IR)+CPHI2*C(IR)
    B(IR)=B(IR)+CPHI3*C(IR)
    C(IR)=C(IR)*CFHII
^{*}
   step size is changed from smaller step size DELY3 to larger step
*
   size DELY4
Ж.
    DO 161 K=IR2,N
    FROD=V(K)*DY4
    A(K) = -DYC4 - PROD
    B(K)=U(K)*DELX+2.0*DYC4+DT
```

```
C(K)=-DYC4+PROD
161 CONTINUE
    DO 170 K=1,N
    D(K)=U(K)*PCON(K,L)*DELX+CONI(K,L)*nT
170 CONTINUE
    D(1)=D(1)-A(1)*CONO
    CALL TRIDI (A,B,C,CON,D,N)
*
ж
  calculation of C's (concentration at variousY-locations is complete
* whether solutions for U's, V's, Theta's, C's during iterative procedure
* are conversed or not is checked. if wes we move to next X-location
* downstream. if not the values calculated in this iteration are the
* suess values for the next iteration. underrelaxation is employed
* refer to section 3.3 for details
ж
    CALL MAX(AU,U,N,EPS)
    IF (EPS.LT.SMALL)GO TO 500
800 DO 120 K=1*N
    AU(K) = AU(K) + ALP*(U(K) - AU(K))
    AV(K)=AV(K)+ALP*(V(K)-AV(K))
    ATHET(K)=ATHET(K)+ALP*(THET(K)-ATHET(K))
    ACON(K)=ACON(K)+ALP*(CON(K)-ACON(K))
120 CONTINUE
    GO TO 400
500 CALL MAX (AU, U, N, EPS)
    TF(EPS.GT.SMALL)GO TO 800
    CALL MAX (ATHET, THET, N, EPS)
    IF (EPS.GT.SMALL)GO TO 800
    CALL MAX(ACON, CON, N, EPS)
    IF(EPS.GT.SMALL)GO TO SOO
*
   the instantaneous local Nusselt and Sherwood numbers are calculated
*
*
   solving the equations (3.13a) and (3.13b) respectively
*
    NUX(L+1) = (3.0-4.0*THET(1)+THET(2))*DY1
    SHX(L+1)=(3.0-4.0%CON(1)+CON(2))*DY1
_{x}
    writing out the values of U,V,Theta and C at every Y-location but
^{*}
*
    at X = 1.0 i.e. at the upper edge of the plate
1/2
    IF (J.NE.50) GO TO 786
    WRITE (25,97)X,TAU,ALP,ITR
    FORMAT(2X, 'X=', F3.1, 1X, 'TIME = ', F4.2, 1X, 'RELAX FACTOR = ', F3.1
97
```

```
D(K)=U(K)*PCON(K,L)*DELX+CONI(K,L)*DT
  CONTINUE
  D(1)=D(1)-A(1)*CONO
  CALL TRIDI (A, B, C, CON, D, N)
lculation of C's (concentration at variousY-locations is complete
ther solutions for U's, V's, Theta's, C's during iterative procedure
conversed or not is checked. if wes we move to next X-location
nstream. if not the values calculated in this iteration are the
ss values for the next iteration. underrelaxation is amployed
er to section 3.3 for details
  CALL MAX(AU,U,N,EPS)
  IF (EPS.LT.SMALL)GO TO 500
  DO 120 K=1,N
  AU(K) = AU(K) + ALP*(U(K) - AU(K))
  AU(K) = AU(K) + ALP*(U(K) - AU(K))
  ATHET(K) = ATHET(K) + ALP*(THET(K) - ATHET(K))
  ACON(K) = ACON(K) + ALP*(CON(K) - ACON(K))
  CONTINUE
  GO TO 400
  CALL MAX (AU, U, N, EPS)
  IF(EPS.GT.SMALL)GO TO 800
  CALL MAX (ATHET, THET, N, EPS)
  TE (EPS.GT.SMALL)GO TO 800
  CALL MAX(ACON, CON, N, EPS)
  IF(EPS.GT.SMALL)GO TO 800
e instantaneous local Nusselt and Sherwood numbers are calculated
lving the equations (3.13a) and (3.13b) respectively
  NUX(L+1)=(3.0-4.0*THET(1)+THET(2))*DY1
  SHX(L+1)=(3.0-4.0*CON(1)+CON(2))*DY1
riting out the values of U,V,Theta and C at every Y-location but
\mathsf{t} \; \mathsf{X} = \mathsf{1.0} \; \mathsf{i.e.} \; \mathsf{a.t.} \; \mathsf{the} \; \mathsf{upper} \; \mathsf{edse} \; \mathsf{of} \; \mathsf{the} \; \mathsf{plate}
  IF (J.NE.50) GO TO 786
  WRITE (25,97)X,TAU,ALP,ITR
  FORMAT(2X, 'X=',F3.1,1X,'TIME = ',F4.2,1X,'RELAX FACTOR = ',F3.1
  1,1X,'ITR = (,12,/)
```

C(K)=-DYC4+PROD

CONTINUE DO 170 K=1,N

```
WRITE(25,98)
   FORMAT(6X, 'K', 4X, 'Y', 5X, 'U', 6X, 'V', 7X, 'THET', 5X, 'CON')
   WRITE(25,93)
   FORMAT(2X,45(1H-))
   WRITE(25,99)(K_0Y(K)_0U(K)_0V(K)_0THET(K)_0CON(K)_0K=1_0N)
   FORMAT(I7,F6.2,F8.4,F8.4,F8.4,F8.4)
   DO 130 K=1,N
   AU(K)=U(K) AU(K)=U(K) ATMET(K)=THET(K) ACON(K)=CON(K)
   PU(K,L+1)=U(K))PTHET(K,L+1)=THET(K))PCON(K,L+1)=CON(K)
   CONTINUE
   CONTINUE
olution has been calculated out till the upper leading edge. now
ie instantaneous mean Nusselt and Sherwood numbers are calculated
olving the eans. (3.16a) and (3.16b) respectively
   NUM=-NUX(M+1)
   SHM=-SHX(M+1)
   DO 132 L=2,M,2
   NUM=NUM+4.0*NUX(L)+2.0*NUX(L+1)
   SHM=SHM+4.0*SHX(L)+2.0*SHX(L+1)
   CONTINUE
   D6=1./(3.*DELX)
   NUM=NUM*D6
   SHM=SHM*D6
   WRITE(25,133)NUM ,SHM
   FORMAT(/,2X, 'MEAN NUSSELT NO= ',F7,5,2X, 'MEAN SHERWOOD NO= '
   1 y F 7 . 5)
   WRITE(23,889)TAU, NUM, SHM
   FORMAT(3X,F4,2,2X,'\',7X,F6,4,6X,'\',6X,F6,4)
   WRITE(25,89)
   FORMAT(2X,52(1H#))
check whether steady state is reached or not. if yes stop the
lculations, if not so to next time step
   DO 990 L=1,M
   DO 991 K=1,N
   A(K)=UI(K,L)
   B(K) = PU(K_{2}L)
   CONTINUE
   CALL MAX(A,B,N,EPS)
   IF(EPS.GT.SSMALL) GO TO 993
   CONTINUE
```

```
\mathbb{C}
   SUBROUTINE TRIDI(A,B,C,X,R,N)
C SOLUTION OF N TRIDIAGONAL TYPE EQUATIONS
   A \times X(J-1) + B \times X(J) + C \times X(J+1) = R
                                               WHERE
C A IS WRITTEN FOR A(J)...,R FOR R(J).X IS THE SOLN VECTOR.
C ALL THE DIMENSIONED VARIABLES HAVE DIMENSION N. HOWEVER,
C A(1)&C(N) ARE NOT DEFINED IN A TRIDIAGONAL SET. VECTORS A
C & R ARE DESTROYED.
    REAL A(N),B(N),C(N),X(N),R(N),BN
    A(N) = A(N) / B(N)
    R(N) = R(N) / B(N)
    DO 10 I=3,N
    K=N+3-I
    J=K-1
    BN=1./(B(J)-A(K)*C(J))
    A(J) = A(J) \times BN
       R(J) = (R(J) - C(J) \times R(K)) \times BN
    X(1) = (R(1) - C(1) \times R(2)) / (B(1) - A(2) \times C(1))
    DO 20 I=2,N
       X(I)=R(I)-A(I)*X(I-1)
   20
    RETURN
SUBROUTINE MAX(X,Y,N,EPS)
    DIMENSION X(N),Y(N)
    EPS=0.0
    DO 10 I=1 vN
    EFSL = ABS(X(I) - Y(I))
    EPS=AMAX1(EPS,EPSL)
10
    CONTINUE
    RETURN
```

END

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